

RESEARCH ARTICLE

WILEY

On the moment dynamics of stochastically delayed linear control systems

Henrik T. Sykora¹  | Mehdi Sadeghpour² | Jin I. Ge³ | Dániel Bachrathy¹  | Gábor Orosz^{4,5} 

¹Department of Applied Mechanics, Budapest University of Technology and Economics, Budapest, Hungary

²Department of Biosciences, Rice University, Houston, Texas, USA

³Department of Computing and Mathematical Sciences, California Institute of Technology, Pasadena, California, USA

⁴Department of Mechanical Engineering, University of Michigan, Ann Arbor, Michigan, USA

⁵Department of Civil and Environmental Engineering, University of Michigan, Ann Arbor, Michigan, USA

Correspondence

Henrik T. Sykora, Department of Applied Mechanics, Budapest University of Technology and Economics, Budapest H-1111, Hungary.
Email: sykora@mm.bme.hu

Funding information

Budapesti Műszaki és Gazdaságtudományi Egyetem, Grant/Award Number: BME FIKP-MI; National Research, Development and Innovation Fund, Grant/Award Numbers: NKFIH ÚNKP-18-3-I-BME-160, TUDFO/51757/2019-ITM; Országos Tudományos Kutatási Alapprogramok, Grant/Award Numbers: FK-124462, PD-124646

Summary

In this article, the dynamics and stability of a linear system with stochastic delay and additive noise are investigated. It is assumed that the delay value is sampled periodically from a stationary distribution. A semi-discretization technique is used to time-discretize the system and derive the mean and second-moment dynamics. These dynamics are used to obtain the stationary moments and the corresponding necessary and sufficient stability conditions. The application of the proposed method is illustrated through the analysis of the Hayes equation with stochastic delay and additive noise. The method is also applied to the control design of a connected automated vehicle. These examples illuminate the effects of stochastic delays on the robustness of dynamical systems.

KEYWORDS

connected automated vehicle, moment stability, stochastic delay, stochastic excitation

1 | INTRODUCTION

Time delays in the control loops can cause instabilities and lead to unwanted oscillations. Analysing the corresponding delay differential equations is necessary to design control algorithms.¹ Time delay-induced vibrations may occur in many applications, including traffic dynamics,² population dynamics,³ gene regulatory networks,⁴ and machine tool

This is an open access article under the terms of the Creative Commons Attribution License, which permits use, distribution and reproduction in any medium, provided the original work is properly cited.

© 2020 The Authors. *International Journal of Robust and Nonlinear Control* published by John Wiley & Sons, Ltd.

vibrations.⁵⁻⁷ When the system parameters (including the delays) are constant and no external excitation occurs, there are well-established methods to investigate the linear as well as the nonlinear dynamics of these systems.⁸⁻¹³ With added noise, stability investigations become more challenging and the stationary motion of a system needs to be carefully characterized.¹⁴⁻¹⁷ Moreover, the system parameters (including the delays) may also vary stochastically requiring sophisticated mathematical tools for stability analysis.¹⁸⁻²⁰ In this article, control systems are considered where both effects are present: the time delays vary stochastically while the system is excited by additive noise.

For example, in vehicular traffic, the driver reaction time typically varies stochastically,²¹ while the additive noise comes from the other vehicles whose motion the driver needs to respond to. In network control systems, delays may vary stochastically due to packet drops, and in the meantime agents need to respond to noisy environment.^{18,22-27} In complex biological networks, like those within cells, external noise is ubiquitous and stochastic delays may be used to model a sequence of reactions.^{28,29} In machining applications, in order to suppress chatter, one may stochastically vary the spindle speed,³⁰ which leads to stochastic delays in the corresponding mathematical model.

One of the most important performance measures for control systems is stability. For stochastic systems, there exist multiple stability definitions. For example, almost sure exponential stability characterizes the mean decay rate of the exponential envelope of the solution of the system.³¹⁻³³ However, this approach does not guarantee the existence of a stationary solution in the presence of additive noise since there can be a finite number of “bursts” as time evolves. A more conservative approach is to consider the first moment (or mean) and second moment (or mean square) stability. This allows one to determine whether the system has bounded stationary solutions or so-called stochastic coherence resonance^{15,16} occurs due to additive noise. Thus, this approach can be utilized to characterize the robustness of the control system.³⁴

For stochastic systems without stochastic time delays, earlier works^{14,15} investigated not only the stability of the system but also the effects of external noise near critical parameters. In particular, small perturbation of a critical parameter were utilized for small degree-of-freedom dynamical systems. The moment stability and stationary second moment of larger stochastic systems has been considered¹⁷ while the time delays were assumed to be constant. Incorporating stochastic delay variations lead to switching systems, whose analysis require sophisticated mathematical approaches. For example, sufficient conditions for second moment stability for switching systems with multiplicative noise were derived.³⁵ Also, necessary and sufficient conditions were given in the presence switching delays.^{4,20} However, the sensitivity against external excitations was not investigated. The stationary behavior was considered for a system with additive noise in optimal control designs^{19,36} and a model predictive control design³⁷ as part of the cost function. However, the resulting controllers are optimal with respect to the prescribed cost function only without leaving any room for more general analysis that is needed for a general control design.

In this work, the stability and stationary solution of continuous time systems with stochastic time delays and additive noise are discussed, without requiring the independence of the two different stochastic excitations. The resulting stochastic dynamical system is approximated using semi-discretization,^{4,17,38} and the corresponding discrete time stochastic map is determined analytically. This map is utilized to obtain the first and second moment dynamics, which in turn, allow us to derive necessary and sufficient conditions for the first and second moment stability and to calculate the stationary first and second moments. The effects of the stochastic delays on the moment dynamics are illustrated through a simple system, called the Hayes equation, where the moments are calculated along the delay-interval for different values of the system parameters (including the holding time). The obtained methods are also applied to the control design of a connected automated vehicle (CAV) subject to packet losses while responding to the noisy motion of its human-driven predecessor.

The article is organized as follows. In Section 2, the semi-discretization is applied to the stochastically delayed system with additive noise and the first and second moment maps are constructed. The computation of the stationary first and second moments of systems driven by white and delay-induced additive noise are discussed in Sections 3 and 4, respectively. In Section 5, some properties of the stationary moments are highlighted through the stochastically delayed Hayes equation, while in Section 6, a control design for a CAV is presented that is robust against stochastic delays. Finally, conclusions are drawn in Section 7.

2 | MODELING AND DISCRETIZATION

In this article, linear systems with stochastic delay and additive noise are considered:

$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}\mathbf{x}(t - \tau(t)) + \boldsymbol{\gamma}(t), \quad (1)$$

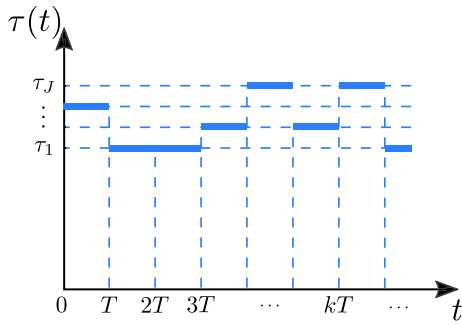


FIGURE 1 A sample realization of the delay process $\tau(t)$ [Colour figure can be viewed at wileyonlinelibrary.com]

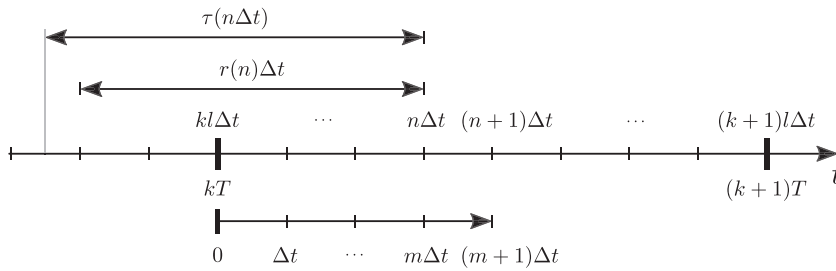


FIGURE 2 Sketch of the time discretization used for the construction of the stochastic map

where the dot indicates the derivative with respect to time t , $\mathbf{x} \in \mathbb{R}^d$ is the d -dimensional state vector, $\tau(t)$ represents a stochastically varying time-delay, $\mathbf{A}, \mathbf{B} \in \mathbb{R}^{d \times d}$ are the coefficient matrices, and $\boldsymbol{\gamma}(t) \in \mathbb{R}^d$ is an arbitrary stochastic noise process with an appropriate probability space. We emphasize that $\boldsymbol{\gamma}(t)$ and $\tau(t)$ are not required to be independent for the results presented in this section to be valid.

While no particular assumptions are made for the additive stochastic process $\boldsymbol{\gamma}(t)$, a specific stochastic process is considered for the time delay $\tau(t)$ which is motivated by applications of wireless communication-based control systems. Namely, the delay process $\tau(t)$ is considered to have piecewise constant trajectories. In particular, the delay is assumed to stay constant for a holding time T before potentially taking on a new value from a finite set $\{\tau_1, \tau_2, \dots, \tau_J\}$, such that $\tau_1 < \tau_2 < \dots < \tau_J$; see Figure 1 for a sample realization of this process. The delays are assumed to be independent and identically distributed (IID) across the holding intervals. The probabilities of the delays can be described as

$$\mathbb{P}(\tau(t) = \tau_j) = w_j, \quad \tau_j \in \{\tau_1, \tau_2, \dots, \tau_J\}, \quad (2)$$

while remaining constant for each interval $t \in (kT, (k+1)T]$.

In order to investigate the stability and stationary solution of the delayed system in Equation (1), the continuous time dynamics can be approximated by a discrete one. This discrete approximation can be constructed using full discretization (such as the Euler method) or semi-discretization.^{4,17,38} Since the latter has superior convergence properties, in this article this approach is utilized. The zeroth order semi-discretization is discussed in the main body of the article, while higher order semi-discretization is shown in Appendix A.

Equation (1) can be discretized to:

$$\dot{\mathbf{x}}(t) \approx \mathbf{A}\mathbf{x}(t) + \mathbf{B}\mathbf{x}((n-r(n))\Delta t) + \boldsymbol{\gamma}(t), \quad t \in [n\Delta t, (n+1)\Delta t), \quad (3)$$

where $n \in \mathbb{N}$ counts the time step under the time resolution $\Delta t = T/l$; see Figure 2.

We choose l such that $\Delta t < \tau_j, j = 1, \dots, J$. The discretized delay is given by

$$r(n) = \left\lfloor \frac{\tau(n\Delta t)}{\Delta t} \right\rfloor, \quad (4)$$

where $\lfloor \cdot \rfloor$ denotes the floor operation. Note that $r(n)$ follows a similar stochastic process as $\tau(t)$, that is,

$$\mathbb{P}(r(n) = r_j) = w_j, \quad (5)$$

where $r_j = \left\lfloor \frac{\tau_j}{\Delta t} \right\rfloor, j = 1, \dots, J$; cf. (2). The differential equation (3) can be solved analytically for one period of length Δt :

$$\mathbf{x}((n+1)\Delta t) = e^{\mathbf{A}\Delta t} \mathbf{x}(n\Delta t) + \int_{n\Delta t}^{(n+1)\Delta t} e^{\mathbf{A}((n+1)\Delta t-s)} \mathbf{B} \mathbf{d}s \mathbf{x}((n-r(n))\Delta t) + \int_{n\Delta t}^{(n+1)\Delta t} e^{\mathbf{A}((n+1)\Delta t-s)} \boldsymbol{\gamma}(s) \mathbf{d}s. \quad (6)$$

By defining the augmented vector

$$\mathbf{y}(n) = [\mathbf{x}^\top(n\Delta t), \mathbf{x}^\top((n-1)\Delta t), \dots, \mathbf{x}^\top((n-r_J)\Delta t)]^\top, \quad (7)$$

system (6) can be written in the compact form

$$\mathbf{y}(n+1) = \mathbf{F}(n)\mathbf{y}(n) + \mathbf{f}(n), \quad (8)$$

where the coefficient matrix and the disturbance vector are

$$\mathbf{F}(n) = \begin{bmatrix} 1 & 2 & 3 & \dots & r(n)+1 & \dots & r_J+1 \\ \mathbf{P} & \mathbf{0} & \mathbf{0} & \dots & \mathbf{R} & \dots & \mathbf{0} \\ \mathbf{I} & \mathbf{0} & \mathbf{0} & \dots & \mathbf{0} & \dots & \mathbf{0} \\ \mathbf{0} & \mathbf{I} & \mathbf{0} & \dots & \mathbf{0} & \dots & \mathbf{0} \\ \vdots & \mathbf{0} & \ddots & & & & \vdots \\ \mathbf{0} & \mathbf{0} & \mathbf{0} & \dots & \mathbf{0} & \mathbf{I} & \mathbf{0} \end{bmatrix}, \quad \mathbf{f}(n) = \begin{bmatrix} \mathbf{w}_\gamma(n) \\ \mathbf{0} \\ \vdots \\ \mathbf{0} \end{bmatrix}, \quad (9)$$

and

$$\mathbf{P} = e^{\mathbf{A}\Delta t}, \quad \mathbf{R} = \int_{n\Delta t}^{(n+1)\Delta t} e^{\mathbf{A}((n+1)\Delta t-s)} \mathbf{B} \mathbf{d}s, \quad \mathbf{w}_\gamma(n) = \int_{n\Delta t}^{(n+1)\Delta t} e^{\mathbf{A}((n+1)\Delta t-s)} \boldsymbol{\gamma}(s) \mathbf{d}s. \quad (10)$$

In case of an invertible coefficient matrix \mathbf{A} , the matrix \mathbf{R} can be expressed as $\mathbf{R} = (e^{\mathbf{A}\Delta t} - \mathbf{I})\mathbf{A}^{-1}\mathbf{B}$. Note that in (9), the block matrix \mathbf{R} is in the $(r(n)+1)$ th block-column of matrix $\mathbf{F}(n)$. In other words, the position of the block matrix \mathbf{R} in the first block-row of $\mathbf{F}(n)$ depends on the instantaneous value of the delay: if $\tau(t) = \tau_j$ in the time interval $(kT, (k+1)T]$, then $r(n) = r_j$. In this case, $\mathbf{F}(n)$ can be substituted by \mathbf{F}_j and, based on (5), $\mathbf{F}(n)$ is IID and follows the probability distribution

$$\mathbb{P}(\mathbf{F}(n) = \mathbf{F}_j) = w_j. \quad (11)$$

Note that the delay value $\tau(t)$ does not change during one holding period $T = l\Delta t$; see Figure 1. Therefore, defining the state vector

$$\mathbf{z}(k) = \mathbf{y}(kl), \quad k = 0, 1, 2, \dots \quad (12)$$

for each holding periods and applying (8) iteratively, the system dynamics can be written as

$$\mathbf{z}(k+1) = \mathbf{G}(k)\mathbf{z}(k) + \mathbf{g}(k), \quad (13)$$

where

$$\begin{aligned} \mathbf{G}(k) &= \mathbf{F}(kl)^l, \\ \mathbf{g}(k) &= \sum_{m=0}^{l-1} \mathbf{F}(kl)^{l-1-m} \mathbf{f}(kl+m), \end{aligned} \quad (14)$$

and

$$\mathbb{P}(\mathbf{G}(k) = \mathbf{F}_j^l) = w_j. \quad (15)$$

We emphasize that since the stochastic process $\tau(t)$ generates the process $r(n)$ which generates $\mathbf{F}(n)$ and $\mathbf{G}(k)$, they all share the same probability distribution function that has been defined in (2).

2.1 | Moment dynamics

Here the general expressions are stated for the time evolution of the mean and second moment of system (13) for a general noise term $\gamma(t)$. In the next section, the dynamics are discussed in more detail when $\gamma(t)$ is a Gaussian white noise.

First, note that the random variables $\mathbf{G}(k)$ and $\mathbf{z}(k)$ in (13) are independent. The matrix $\mathbf{G}(k)$ depends only on the delay value in the time interval $kT \leq t \leq (k+1)T$ with probability mass function given by (15), while the vector $\mathbf{z}(k)$ depends on the values of the delay $\tau(t)$ and the noise $\gamma(t)$ in the time interval $0 \leq t \leq kT$. Therefore, given the assumption that the delay switchings are IID, $\mathbf{G}(k)$ and $\mathbf{z}(k)$ are independent. Now by taking the expected value of both sides of Equation (13) and exploiting the independence of $\mathbf{G}(k)$ and $\mathbf{z}(k)$, the mean dynamics

$$\bar{\mathbf{z}}(k+1) = \bar{\mathbf{G}}\bar{\mathbf{z}}(k) + \bar{\mathbf{g}}(k), \quad (16)$$

is obtained, where

$$\begin{aligned} \bar{\mathbf{z}}(k) &:= \mathbb{E}(\mathbf{z}(k)), \\ \bar{\mathbf{g}}(k) &:= \mathbb{E}(\mathbf{g}(k)), \\ \bar{\mathbf{G}} &:= \mathbb{E}(\mathbf{G}(k)) = \sum_{j=1}^J w_j \mathbf{F}_j^l, \end{aligned} \quad (17)$$

and $\mathbb{E}(\cdot)$ denotes the expected value operator.

To obtain the second moment dynamics, the vector

$$\bar{\bar{\mathbf{z}}}(k) := \mathbb{E}(\mathbf{z}(k) \otimes \mathbf{z}(k)), \quad (18)$$

is defined, where \otimes denotes the Kronecker product. Here the fact that $\mathbb{E}(\mathbf{z}(k) \otimes \mathbf{z}(k)) = \text{vec}(\mathbb{E}(\mathbf{z}(k)\mathbf{z}(k)^\top))$ is used, where $\text{vec}(\cdot)$ is the vectorization operator that places the columns of a matrix below each other. The definition using the Kronecker product in (18) is suitable for the second moment dynamics analysis carried out in this article. By the properties of Kronecker product and the independence of $\mathbf{G}(k)$ and $\mathbf{z}(k)$, the second moment dynamics

$$\bar{\bar{\mathbf{z}}}(k+1) = \bar{\bar{\mathbf{G}}}\bar{\bar{\mathbf{z}}}(k) + \bar{\bar{\mathbf{c}}}_z(k) + \bar{\bar{\mathbf{g}}}(k), \quad (19)$$

is derived, where

$$\begin{aligned} \bar{\bar{\mathbf{G}}} &:= \mathbb{E}(\mathbf{G}(k) \otimes \mathbf{G}(k)) = \sum_{j=1}^J w_j \mathbf{F}_j^l \otimes \mathbf{F}_j^l, \\ \bar{\bar{\mathbf{c}}}_z(k) &:= \mathbb{E}((\mathbf{g}(k) \otimes \mathbf{G}(k) + \mathbf{G}(k) \otimes \mathbf{g}(k))\mathbf{z}(k)), \\ \bar{\bar{\mathbf{g}}}(k) &:= \mathbb{E}(\mathbf{g}(k) \otimes \mathbf{g}(k)). \end{aligned} \quad (20)$$

The details of the derivation of the term $\bar{\bar{\mathbf{c}}}_z(k)$ are shown in Appendix B.

In the next section, the specific forms of Equations (17) to (20) are derived, while considering an additive Gaussian white noise.

3 | DYNAMICS AND STABILITY WITH GAUSSIAN WHITE NOISE

So far, no particular type of stochastic process was assumed for the noise $\gamma(t)$ in (13). Thus, (17) to (20) are valid in general. For simplicity, in this section, $\gamma(t)$ is considered as a Gaussian white noise process with the form

$$\gamma(t) = \sum_{i=1}^{d_\xi} \sigma_i \xi_i(t), \quad (21)$$

where ξ_i , $i = 1, \dots, d_\xi$, are mutually independent Gaussian white noise processes with mean $\mathbb{E}(\xi_i(t)) = 0$ and correlation function $\mathbb{E}(\xi_i(t)\xi_j(s)) = \delta(t-s)\delta_{ij}$. The vectors $\sigma_i \in \mathbb{R}^d$, $i = 1, \dots, d_\xi$, denote the noise intensities. The noise process defined in (21) can be written in the compact form

$$\gamma(t) = \Omega \xi(t), \quad (22)$$

where the i th column of the matrix $\Omega \in \mathbb{R}^{d \times d_\xi}$ is σ_i and the i th component of the vector $\xi(t)$ is $\xi_i(t)$. Observe that $\mathbb{E}(\gamma(t)) = \mathbf{0}$. Next, the behavior and stability of the mean dynamics (16) and the second moment dynamics (19) are investigated when $\gamma(t)$ is given by (21).

3.1 | Mean dynamics

In this section, it is illustrated that the mean dynamics (16) can be simplified to

$$\bar{\mathbf{z}}(k+1) = \bar{\mathbf{G}}\bar{\mathbf{z}}(k), \quad (23)$$

by showing that $\bar{\mathbf{g}}(k) = 0$ when the noise $\gamma(t)$ is given by (21).

According to (14) and (16), the additive vector is given as

$$\bar{\mathbf{g}}(k) = \sum_{m=0}^{l-1} \mathbb{E}(\mathbf{F}(kl)^{l-1-m} \mathbf{f}(kl+m)). \quad (24)$$

Note that $\mathbf{F}(kl)$ only depend on the delay $\tau(t)$ in the interval $(kT, (k+1)T]$ while $\mathbf{f}(kl+m)$, $m = 0, 1, \dots, l-1$, depends only on the Gaussian white noise $\xi(t)$ in the interval $(kT + m\Delta t, kT + (m+1)\Delta t]$. Since $\tau(t)$ and $\xi(t)$ are independent $\mathbf{F}(kl)$ and $\mathbf{f}(kl+m)$ are also independent. Thus, (24) becomes

$$\bar{\mathbf{g}}(k) = \sum_{m=0}^{l-1} \mathbb{E}(\mathbf{F}(kl)^{l-1-m}) \mathbb{E}(\mathbf{f}(kl+m)). \quad (25)$$

On the other hand with Gaussian white noise (10) yields

$$\begin{aligned} \mathbb{E}(\mathbf{w}_\gamma(kl+m)) &= \mathbb{E}\left(\int_{(kl+m)\Delta t}^{(kl+m+1)\Delta t} e^{\mathbf{A}((kl+m+1)\Delta t-s)} \Omega d\mathbf{W}_s\right) \\ &= \mathbb{E}\left(\int_0^{\Delta t} e^{\mathbf{A}(\Delta t-s)} \Omega d\mathbf{W}_s\right) \\ &= \mathbf{0}, \end{aligned} \quad (26)$$

where \mathbf{W}_t is the Wiener process vector corresponding to the white noise $\xi(t)$, namely

$$\mathbf{W}_t = \int_0^t \xi(s) ds. \quad (27)$$

The last equality in (26) follows from the zero mean property of the Itô integral.³⁹ Now according to the definition (9), we have

$$\mathbb{E}(\mathbf{f}(kl+m)) = \mathbf{0}, \quad (28)$$

and hence (25) yields $\bar{\mathbf{g}}(k) = 0$ and the mean dynamics are given by (23).

From system (23), it is concluded that the mean $\bar{\mathbf{z}}(k)$ converges to $\mathbf{0}$ as $k \rightarrow \infty$, if and only if

$$\rho\left(\bar{\mathbf{G}} = \sum_{j=1}^J w_j \mathbf{F}_j^l\right) < 1, \quad (29)$$

where $\rho(\cdot)$ denotes the spectral radius, namely

$$\rho(\bar{\mathbf{G}}) = \max_z \{ \text{abs}(z), z \in \mathbb{C} : \det(\bar{\mathbf{G}} - z\mathbf{I}) = 0 \}. \quad (30)$$

3.2 | Second moment dynamics

In this subsection, the second moment dynamics (19) is simplified to the following form

$$\bar{\bar{\mathbf{z}}}(k+1) = \bar{\bar{\mathbf{G}}}\bar{\bar{\mathbf{z}}}(k) + \bar{\bar{\mathbf{g}}}, \quad (31)$$

and the disturbance term $\bar{\bar{\mathbf{g}}}$ is calculated explicitly. To do this, first the term $\bar{\bar{\mathbf{c}}}_z(k)$ in (19) is considered. Note that similar to the independence of $\mathbf{z}(k)$ and $\mathbf{G}(k)$ that was described at the beginning of Section 2.1, $\mathbf{z}(k)$ is also independent of $\mathbf{g}(k)$ in view of (14). Therefore, from (20), it follows that

$$\bar{\bar{\mathbf{c}}}_z(k) = \mathbb{E}(\mathbf{g}(k) \otimes \mathbf{G}(k) + \mathbf{G}(k) \otimes \mathbf{g}(k))\mathbb{E}(\mathbf{z}(k)). \quad (32)$$

Now observe that

$$\begin{aligned} \mathbb{E}(\mathbf{g}(k) \otimes \mathbf{G}(k)) &= \mathbb{E} \left(\left(\sum_{m=0}^{l-1} \mathbf{F}(kl)^{l-1-m} \mathbf{f}(kl+m) \right) \otimes \mathbf{G}(k) \right) \\ &= \sum_{m=0}^{l-1} \mathbb{E}((\mathbf{F}(kl)^{l-1-m} \otimes \mathbf{G}(k))(\mathbf{f}(kl+m) \otimes \mathbf{I})) \\ &= \sum_{m=0}^{l-1} \mathbb{E}((\mathbf{F}(kl)^{l-1-m} \otimes \mathbf{G}(k))\mathbb{E}(\mathbf{f}(kl+m) \otimes \mathbf{I})) \\ &= \mathbf{0}. \end{aligned} \quad (33)$$

In the second line above, the mixed-product property of the Kronecker product was used and \mathbf{I} denotes the identity matrix, while the last equality comes from (28). Similar to (33), it can be shown that $\mathbb{E}(\mathbf{G}(k) \otimes \mathbf{g}(k)) = \mathbf{0}$, and therefore, (32) leads to

$$\bar{\bar{\mathbf{c}}}_z(k) = \mathbf{0}. \quad (34)$$

Finally, the last term $\bar{\bar{\mathbf{g}}}(k)$ in (19) is expressed as:

$$\begin{aligned} \bar{\bar{\mathbf{g}}}(k) &= \mathbb{E}(\mathbf{g}(k) \otimes \mathbf{g}(k)) \\ &= \mathbb{E} \left(\left(\sum_{m=0}^{l-1} \mathbf{F}(kl)^{l-1-m} \mathbf{f}(kl+m) \right) \otimes \left(\sum_{m=0}^{l-1} \mathbf{F}(kl)^{l-1-m} \mathbf{f}(kl+m) \right) \right). \end{aligned} \quad (35)$$

Using the definition of $\mathbf{w}_\gamma(n)$ from (10) considering (22), we obtain

$$\begin{aligned} \mathbf{f}(kl+m) &= \begin{bmatrix} \mathbf{w}_\gamma(kl+m) \\ \mathbf{0} \\ \vdots \\ \mathbf{0} \end{bmatrix} \\ &= \begin{bmatrix} \int_0^{\Delta t} e^{\mathbf{A}(\Delta t-s)} \boldsymbol{\Omega} d\mathbf{W}_s \\ \mathbf{0} \\ \vdots \\ \mathbf{0} \end{bmatrix}. \end{aligned} \quad (36)$$

By defining

$$\mathbf{r}_{w,m} := \int_0^{\Delta t} e^{\mathbf{A}(\Delta t-s)} \mathbf{\Omega} d\mathbf{W}_s, \quad \mathbf{f}_{w,m} := \begin{bmatrix} \mathbf{r}_{w,m} \\ \mathbf{0} \\ \vdots \\ \mathbf{0} \end{bmatrix}, \quad (37)$$

(35) can be written as

$$\begin{aligned} \bar{\mathbf{g}}(k) &= \mathbb{E} \left(\left(\sum_{m=0}^{l-1} \mathbf{F}(kl)^{l-1-m} \mathbf{f}_{w,m} \right) \otimes \left(\sum_{m=0}^{l-1} \mathbf{F}(kl)^{l-1-m} \mathbf{f}_{w,m} \right) \right) \\ &= \mathbb{E} \left(\sum_{m=0}^{l-1} \sum_{m'=0}^{l-1} (\mathbf{F}(kl)^{l-1-m} \otimes \mathbf{F}(kl)^{l-1-m'}) (\mathbf{f}_{w,m} \otimes \mathbf{f}_{w,m'}) \right) \\ &= \sum_{m=0}^{l-1} \sum_{m'=0}^{l-1} \mathbb{E}(\mathbf{F}(kl)^{l-1-m} \otimes \mathbf{F}(kl)^{l-1-m'}) \mathbb{E}(\mathbf{f}_{w,m} \otimes \mathbf{f}_{w,m'}). \end{aligned} \quad (38)$$

Note that the terms $\mathbf{f}_{w,m}$ and $\mathbf{f}_{w,m'}$ are independent for $m \neq m'$ because of the independent increments property of the Wiener process.³⁹ Thus,

$$\mathbb{E}(\mathbf{f}_{w,m} \otimes \mathbf{f}_{w,m'}) = \mathbb{E}(\mathbf{f}_{w,m}) \otimes \mathbb{E}(\mathbf{f}_{w,m'}) = \mathbf{0}, \text{ for } m \neq m'. \quad (39)$$

Therefore, (38) can be reduced to

$$\bar{\mathbf{g}}(k) = \sum_{m=0}^{l-1} \mathbb{E}(\mathbf{F}(kl)^{l-1-m} \otimes \mathbf{F}(kl)^{l-1-m}) \mathbb{E}(\mathbf{f}_{w,m} \otimes \mathbf{f}_{w,m}). \quad (40)$$

Furthermore, using (37), we obtain

$$\mathbb{E}(\mathbf{f}_{w,m} \otimes \mathbf{f}_{w,m}) = \mathbb{E} \left(\begin{bmatrix} \mathbf{r}_{w,m} \\ \mathbf{0} \\ \vdots \\ \mathbf{0} \end{bmatrix} \otimes \begin{bmatrix} \mathbf{r}_{w,m} \\ \mathbf{0} \\ \vdots \\ \mathbf{0} \end{bmatrix} \right) = \text{vec} \left(\begin{bmatrix} \mathbf{Q} & \mathbf{0} & \dots & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \dots & \mathbf{0} \\ \vdots & \vdots & \ddots & \vdots \\ \mathbf{0} & \mathbf{0} & \dots & \mathbf{0} \end{bmatrix} \right), \quad (41)$$

where

$$\mathbf{Q} = \mathbb{E}(\mathbf{r}_{w,m} \mathbf{r}_{w,m}^\top) = \mathbb{E} \left(\int_0^{\Delta t} e^{\mathbf{A}(\Delta t-s)} \mathbf{\Omega} d\mathbf{W}_s \int_0^{\Delta t} e^{\mathbf{A}(\Delta t-s)} \mathbf{\Omega} d\mathbf{W}_s \right). \quad (42)$$

Using the Itô isometry³⁹

$$\mathbb{E} \left(\int_0^{\Delta t} f(t) d\mathbf{W}_t \int_0^{\Delta t} g(t) d\mathbf{W}_t \right) = \int_0^{\Delta t} f(t) g(t) dt, \quad (43)$$

one can simplify the stochastic integral (42) to the deterministic one

$$\mathbf{Q} = \int_0^{\Delta t} e^{\mathbf{A}(\Delta t-t)} \mathbf{\Omega} \mathbf{\Omega}^\top e^{\mathbf{A}^\top(\Delta t-t)} dt. \quad (44)$$

Now, by defining

$$\hat{\mathbf{F}}_j := \sum_{m=0}^{l-1} (\mathbf{F}_j^{l-1-m} \otimes \mathbf{F}_j^{l-1-m}) = \mathbf{I} \otimes \mathbf{I} + \mathbf{F}_j \otimes \mathbf{F}_j + \dots + \mathbf{F}_j^{l-1} \otimes \mathbf{F}_j^{l-1}, \quad \bar{\mathbf{f}}_w := \text{vec} \begin{pmatrix} \mathbf{Q} & \mathbf{0} & \dots & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \dots & \mathbf{0} \\ \vdots & \vdots & \ddots & \vdots \\ \mathbf{0} & \mathbf{0} & \dots & \mathbf{0} \end{pmatrix}, \quad (45)$$

(40) can be reduced to

$$\begin{aligned} \bar{\mathbf{g}}(k) &= \sum_{j=1}^J w_j \sum_{m=0}^{l-1} (\mathbf{F}_j^{l-1-m} \otimes \mathbf{F}_j^{l-1-m}) \bar{\mathbf{f}}_w \\ &= \left(\sum_{j=1}^J w_j \hat{\mathbf{F}}_j \right) \bar{\mathbf{f}}_w \\ &:= \bar{\bar{\mathbf{g}}}. \end{aligned} \quad (46)$$

That is, the second moment dynamics originally defined in (19) are given by the second moment map (31) since $\bar{\mathbf{c}}_z(k)$ (cf. (34)) where the disturbance term $\bar{\mathbf{g}}$ is given by (46).

System (31) converges to the stationary solution

$$\bar{\mathbf{z}}^* = (\mathbf{I} - \bar{\mathbf{G}})^{-1} \bar{\mathbf{g}}, \quad (47)$$

as $k \rightarrow \infty$ if and only if

$$\rho \left(\bar{\mathbf{G}} = \sum_{j=1}^J w_j \mathbf{F}_j^l \otimes \mathbf{F}_j^l \right) < 1. \quad (48)$$

4 | DYNAMICS AND STABILITY WITH GAUSSIAN WHITE NOISE AND DELAY-INDUCED NOISE

In this section, generalizations of the results obtained in the previous section are provided in the case of when the noise is given by

$$\boldsymbol{\gamma}(t) = \boldsymbol{\Omega} \boldsymbol{\xi}(t) + \boldsymbol{\kappa}(\tau(t) - \bar{\tau}), \quad (49)$$

where $\bar{\tau} = \sum_{j=1}^J w_j \tau_j$ is the mean delay and $\boldsymbol{\kappa}$ is a constant vector; cf. (22). This generalization is motivated by the fact that in some applications delay fluctuations may result in an additive noise term in the dynamics. This situation is further discussed in Section 6 where such delay-induced noise term arises in the control design of a connected automated vehicle (CAV).

When the noise $\boldsymbol{\gamma}(t)$ is given by (49), the mean dynamics are described by

$$\bar{\mathbf{z}}(k+1) = \bar{\mathbf{G}} \bar{\mathbf{z}}(k) + \bar{\mathbf{g}}, \quad (50)$$

where

$$\bar{\mathbf{g}} = \sum_{j=1}^J w_j \hat{\mathbf{F}}_j \bar{\mathbf{f}}_j, \quad (51)$$

and

$$\begin{aligned} \hat{\mathbf{F}}_j &= \sum_{m=0}^{l-1} \mathbf{F}_j^{l-1-m} = \mathbf{I} + \mathbf{F}_j + \dots + \mathbf{F}_j^{l-1}, \\ \bar{\mathbf{f}}_j &= [((\mathbf{e}^{\mathbf{A}\Delta t} - \mathbf{I})\mathbf{A}^{-1} \boldsymbol{\kappa}(\tau_j - \bar{\tau}))^\top, \mathbf{0}^\top, \dots, \mathbf{0}^\top]^\top. \end{aligned} \quad (52)$$

Compared to (23), here the disturbance term $\bar{\mathbf{g}}$ arises in the mean dynamics which is a result of the additive noise term in (49) due to the delay stochasticity. The details of the derivation of the disturbance term $\bar{\mathbf{g}}$ are shown in Appendix C.

From system (50), it can be concluded that the mean $\bar{\mathbf{z}}(k)$ converges to the equilibrium

$$\bar{\mathbf{z}}^* = (\mathbf{I} - \bar{\mathbf{G}})^{-1} \bar{\mathbf{g}}, \quad (53)$$

as $k \rightarrow \infty$, if and only if (29) holds. That is, the stability condition for the mean is the same as in the previous section. The difference is that when the noise is given by (49), the equilibrium mean is nonzero and given by (53).

The second moment dynamics, for the noise $\boldsymbol{\gamma}(t)$ given by (49), are described by

$$\bar{\bar{\mathbf{z}}}(k+1) = \bar{\bar{\mathbf{G}}} \bar{\bar{\mathbf{z}}}(k) + \bar{\bar{\mathbf{H}}} \bar{\mathbf{z}}(k) + \bar{\bar{\mathbf{g}}}_2, \quad (54)$$

where

$$\bar{\bar{\mathbf{H}}} := \sum_{j=1}^J w_j (\hat{\mathbf{F}}_j \bar{\mathbf{f}}_j \otimes \mathbf{F}_j^l + \mathbf{F}_j^l \otimes \hat{\mathbf{F}}_j \bar{\mathbf{f}}_j), \quad (55)$$

and

$$\begin{aligned} \bar{\bar{\mathbf{g}}}_2 &= \left(\sum_{j=1}^J w_j \hat{\mathbf{F}}_j \right) \bar{\mathbf{f}}_w + \sum_{j=1}^J w_j (\hat{\mathbf{F}}_j \otimes \hat{\mathbf{F}}_j) (\bar{\mathbf{f}}_j \otimes \bar{\mathbf{f}}_j) \\ &= \bar{\mathbf{g}} + \sum_{j=1}^J w_j (\hat{\mathbf{F}}_j \otimes \hat{\mathbf{F}}_j) (\bar{\mathbf{f}}_j \otimes \bar{\mathbf{f}}_j). \end{aligned} \quad (56)$$

The derivations of (54) to (56) are detailed in Appendix C.

Note that the difference between (54) and (31) is, that in (54), an additional term $\bar{\bar{\mathbf{H}}} \bar{\mathbf{z}}(k)$ appears which is a result of the interaction between the Gaussian noise and the delay-induced noise. Also, the disturbance term $\bar{\bar{\mathbf{g}}}_2$ differs from $\bar{\mathbf{g}}$ as it has an extra term stemming from delay-induced noise; cf. (45) and (56).

Combining systems (50) and (54), one can write

$$\begin{bmatrix} \bar{\mathbf{z}}(k) \\ \bar{\bar{\mathbf{z}}}(k+1) \end{bmatrix} = \begin{bmatrix} \bar{\mathbf{G}} & \mathbf{0} \\ \bar{\bar{\mathbf{H}}} & \bar{\bar{\mathbf{G}}} \end{bmatrix} \begin{bmatrix} \bar{\mathbf{z}}(k) \\ \bar{\bar{\mathbf{z}}}(k) \end{bmatrix} + \begin{bmatrix} \bar{\mathbf{g}} \\ \bar{\bar{\mathbf{g}}}_2 \end{bmatrix}, \quad (57)$$

which has the stationary solution

$$\begin{aligned} \bar{\mathbf{z}}^* &= (\mathbf{I} - \bar{\mathbf{G}})^{-1} \bar{\mathbf{g}}, \\ \bar{\bar{\mathbf{z}}}^* &= (\mathbf{I} - \bar{\bar{\mathbf{G}}})^{-1} (\bar{\bar{\mathbf{g}}}_2 + \bar{\bar{\mathbf{H}}} \bar{\mathbf{z}}^*), \end{aligned} \quad (58)$$

if and only if both (29) and (48) hold. That is, the condition for the stationary moments are the same as in the previous section. However, the stationary mean is not zero, while the stationary second moment is different, cf. (47) and (58).

5 | AN ILLUSTRATIVE EXAMPLE

To illustrate the dynamics and stability analysis of the mean and the second moment established in the previous sections, the scalar Hayes equation with stochastic delay is considered. Assuming additive noise that is the sum of a Gaussian noise and a delay-induced noise, we obtain

$$\dot{x}(t) = ax(t) + bx(t - \tau(t)) + \sigma \xi(t) + \kappa(\tau(t) - \bar{\tau}), \quad (59)$$

(cf. (49)) where $a, b, \sigma, \kappa \in \mathbb{R}$. The delay $\tau(t)$ is assumed to take one of the values $\tau_1 = 0.2$, $\tau_2 = 0.3$, or $\tau_3 = 0.4$ at each holding period with equal probability $w_j = 1/3$, $j = 1, 2, 3$ resulting the mean delay $\bar{\tau} = \sum_{j=1}^3 w_j \tau_j = 0.3$.

To investigate the stability properties and the stationary first and second moments of (59), the system (57) is constructed. The condition (29) is used to calculate mean stability, while (48) is utilized for second moment stability. The stationary mean and second moment are given in (58). The scalar versions of the terms that appear in conditions (29) and (48) and (58) are provided below.

Here we have the matrices

$$\mathbf{F}_j = \begin{bmatrix} 1 & 2 & \dots & r_j+1 & \dots & r_j+1 \\ e^{a\Delta t} & 0 & \dots & \frac{b}{a}(e^{a\Delta t} - 1) & \dots & 0 \\ 1 & 0 & 0 & \dots & 0 & 0 \\ 0 & 1 & 0 & \dots & 0 & 0 \\ 0 & 0 & 1 & \dots & 0 & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots & \vdots \\ 0 & 0 & 0 & \dots & 1 & 0 \end{bmatrix}, \quad (60)$$

where $r_j = \lfloor \tau_j / \Delta t \rfloor$ and $r_J = \lfloor \tau_J / \Delta t \rfloor$; cf.(9). Furthermore, (52) yields

$$\bar{\mathbf{f}}_j = \left[\frac{\kappa}{a}(e^{a\Delta t} - 1)(\tau_j - \bar{\tau}), 0, \dots, 0 \right]^\top, \quad (61)$$

and from (37), $\mathbf{f}_{w,m}$ can be derived as

$$\mathbf{f}_{w,m} = [r_{w,m}, 0, \dots, 0]^\top, \quad \text{where } r_{w,m} = \sigma \int_0^{\Delta t} e^{a(\Delta t-s)} dW_s. \quad (62)$$

Finally, (45) becomes

$$\bar{\mathbf{f}}_w = [Q, 0, \dots, 0]^\top, \quad \text{where } Q = \sigma^2 \int_0^{\Delta t} e^{2a(\Delta t-t)} dt = \frac{\sigma^2}{2a}(e^{2a\Delta t} - 1). \quad (63)$$

Given the terms in (60) to (63), to use condition (29), one can obtain $\bar{\mathbf{G}}$ from (17), $\bar{\bar{\mathbf{G}}}$ from (20). For the stationary mean and second moments (58), one can obtain $\bar{\mathbf{g}}$ from (51) and (52), $\bar{\bar{\mathbf{g}}}_2$ from (45), (52) and (56), and $\bar{\bar{\mathbf{H}}}$ from (55). To generate the results below the time step $\Delta t = 0.01$ is used. This is sufficiently small to approximate well the stability boundaries of this example (see also the discussion in Reference 28 where this example is taken from). To determine the mean and second moment stability, conditions (29) and (48) are used, that is, if $\rho(\bar{\mathbf{G}}) < 1$ and $\rho(\bar{\bar{\mathbf{G}}}) < 1$ hold, then the system is mean and second moment stable, respectively.

The left panel in Figure 3 shows the stability boundaries in the (a, b) parameter space for holding times $T = 0.1, 0.3, 0.5$, and 1 as indicated by color. Note that the holding time values are chosen such that they span a relatively large range with respect to the delay values, that is, $T = 0.1$ is smaller than all the delays, $T = 0.3$ is the mean delay, and $T = 0.5$ and 1 are larger than all the delays. Dashed lines indicate mean stability boundaries while solid lines bound the second moment stability regions. That is, on the left side of the dashed lines, the mean converges to the stationary solution (58) while on the right side, it diverges to infinity. Similarly, on the left side of the solid lines, the second moment converges to the stationary solution (58) and it diverges on the right side of these boundaries.

The right panel in Figure 3 shows three different moment realizations to demonstrate the three types of stability states: moment stable, first moment stable-second moment unstable, and moment unstable. In order to illustrate the behaviour of the dynamical system (57) in the different parameter domains, the ensemble standard deviation given by

$$\text{StD}(x) = \sqrt{\mathbb{E}(x^2) - (\mathbb{E}(x))^2}, \quad (64)$$

is used. It can be observed that when system (59) is moment stable (case A), then both the first and second moments converge. However, as the parameters are moved toward the unstable areas first the second moment diverges (case B), then the first moment also loses stability (case C).

FIGURE 3 (Left) Mean (dashed) and second moment (solid) stability boundaries for the Hayes equation (59) for different values of the holding time T as indicated by color. The charts were determined using $\Delta t = 0.01$ and using a bisection method.⁴⁰ (Right) Numerical simulation results for points A, B, C when $T = 1$, with the mean and standard deviations highlighted [Colour figure can be viewed at wileyonlinelibrary.com]

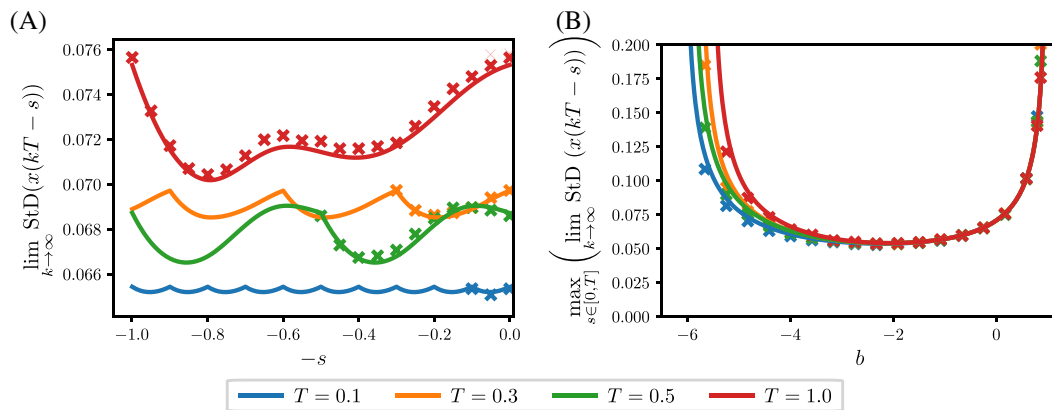
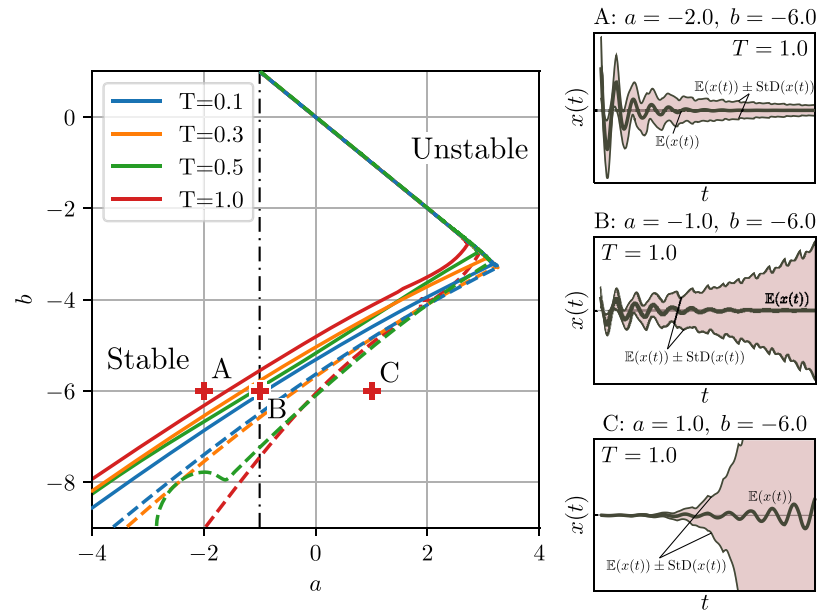


FIGURE 4 A, Stationary standard deviation over a holding period T for the Hayes equation (59) with parameters $a = -1$, $b = -4.5$, and white noise excitation ($\kappa = 0$, $\sigma = 1$). B, The maximum values of the stationary standard deviation over a holding time along the dashed-dotted line in Figure 3. The continuous lines denote the analytical results, while the \times -s are obtained by Monte Carlo simulations [Colour figure can be viewed at wileyonlinelibrary.com]

Next, to validate the analytical result, the stationary mean and standard deviation are calculated utilizing (58) with time resolution $\Delta t = 0.01$, and the results are compared with statistical evaluations of the mean and second moment obtained by Monte Carlo simulations of system (59). For these simulations, we use the Euler-Maruyama method with a time step $\delta t = 0.001$.

In Figure 4A, the stationary standard deviations are shown over as a function of time for different holding times when the additive noise is purely due to the white noise excitation, that is, $\kappa = 0$ and $\sigma = 1$ in (59). The results are shown up to $s = 1$ for all holding times, that is, the augmented vector (7) includes the time history up to the largest holding time $T = 1$. The results of the semi-discretization (solid lines) are compared with the results obtained by Monte Carlo simulations (\times -s). Note that the stationary standard deviation gained by both methods show periodic variation with period T due to the periodic switching of the delay process $\tau(t)$, even though the additive noise gives no periodic excitation.⁴¹

In Figure 4B, we show the maximum value of the stationary standard deviation within a holding interval while setting $a = -1$ and varying b within $[-6.5, 1]$; see the dashed-dotted line in the stability chart in Figure 3. As the parameters of the system approach the stability boundary, the effect of the noise on the stationary standard deviation is magnified, leading to the so-called stochastic coherence resonance.¹⁴⁻¹⁶ Note that the stationary mean is constantly zero for any holding time

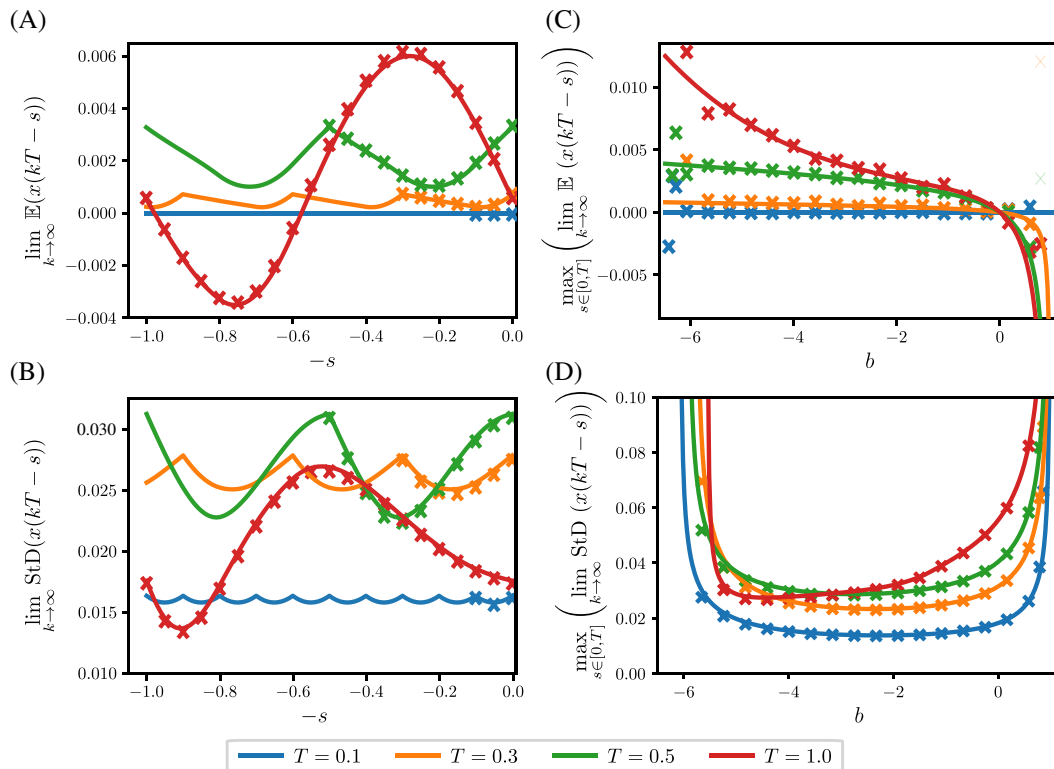


FIGURE 5 A, Stationary mean and, B, standard deviation over a holding period T for the Hayes equation (59) with parameters $a = -1$, $b = -4.5$, and delay excitation ($\kappa = 1$, $\sigma = 0$). Maximum values of, C, the mean and, D, the stationary standard deviation along the dashed-dotted line in Figure 3. The continuous lines denote the analytical results while the x-s denote the results obtained by Monte Carlo simulations [Colour figure can be viewed at wileyonlinelibrary.com]

T . This is due to the fact that the white noise excitation is independent of the random delay fluctuations. This can also be verified by observing that $\bar{\mathbf{g}} = 0$ in (51), because $\bar{\mathbf{f}}_j = 0$ in (61) due to $\kappa = 0$ as shown in Section 3.2.

In Figure 5A,B, the stationary mean and standard deviation are shown as function of time over a holding time interval for the delay-induced noise scenario ($\kappa = 1$ and $\sigma = 0$ in (59)). Note that since the delay $\tau(t)$ switches at every kT , the stationary behavior shows a periodic behavior with period T . This can be observed in both the stationary mean and standard deviation. In Figure 5C,D, the maximum value of the stationary mean and standard deviation over a holding interval are depicted for the delay-induced noise case when considering parameters $a = -1$, $b \in [-6.5, 1]$. In Figure 5D, the stochastic resonance can be observed again for the parameters in the vicinity of the stability boundary. Meanwhile a stability loss can be observed for the stationary first moment in Figure 5C near $b = 1$ (upper stability boundary).

In Figure 6, the maxima of the stationary moments are shown as a function of the holding time T . In Figure 6A, it can be observed that if the holding time T is larger than the smallest delay value τ_1 , the stationary mean is not zero anymore. In particular, examining $\bar{\mathbf{g}}$ in (51), one can see that for $T > \tau_1$ the quantity $\bar{\mathbf{g}}$ is nonzero yielding non-zero stationary mean, even though the excitation term $\kappa(\tau(t) - \bar{\tau})$ has a zero expected value. The stationary standard deviation in Figure 6 does not show any special behaviour with respect to the holding time T around the smallest delay value τ_1 , however, after reaching a maximum it decreases as T increases. This can be due to the reason that as T takes greater values, the delay-induced noise causes resonance-like phenomenon. However, if T is further increased, the same time delay is held for longer time allowing the variations around the temporary equilibrium state to settle before the next additive time delay-induced switching in $\kappa(\tau(t) - \bar{\tau})$ perturbs system (59).

Note that the Monte-Carlo simulations approximate the analytical results well. However, the analytical approach only requires a few matrix multiplications and solving a system of linear equations, which are orders of magnitudes faster than calculating thousands of realizations and statistically evaluating them. This suggests that the proposed method is a very efficient tool to investigate the behavior of such systems, especially for higher dimensional state variables.

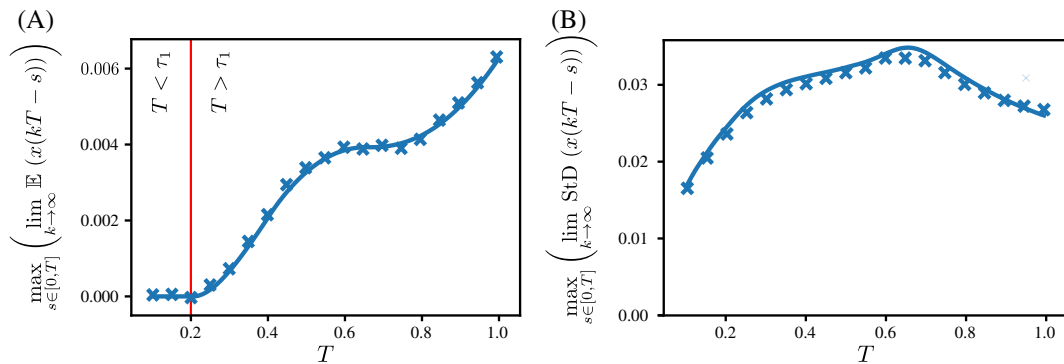


FIGURE 6 Maxima of the, A, stationary mean and, B, standard deviation over a holding period T for the Hayes equation (59) with parameters $a = -1$, $b = -4.5$, and different T holding times for delay excitation ($\kappa = 1$, $\sigma = 0$). The continuous lines denote the analytical results while the x-s denote the results obtained by Monte Carlo simulations. Note that the first moment is zero for $T < \tau_1$ [Colour figure can be viewed at wileyonlinelibrary.com]

To summarize, in this section, the capabilities of the general method were demonstrated on the Hayes equation, namely, that it can be utilized to determine the second moment stability (Figure 3) and the stationary moments (Figures 4 to 6) efficiently. It was shown that the stationary moments show periodic behavior and their maximum values vary with respect the parameters a and b and the holding time T . The results were validated with the help of Monte-Carlo simulations.

6 | STOCHASTIC EFFECTS IN CONNECTED VEHICLE SYSTEMS

In this section, we consider the influence of stochastic time delay and additive noise on connected vehicle systems. In particular, we focus on the longitudinal dynamics of a connected automated vehicle (CAV) following a connected human-driven vehicle (CHV) that broadcasts its GPS position and speed via wireless vehicle-to-vehicle (V2V) communication. When receiving the packets, the CAV can respond to the motion of the CHV by adjusting the throttle or applying the brakes. This is referred as connected cruise control and the effects of time delays in such systems have been investigated both theoretically and experimentally.^{42,43} There are two sources of delay in this example. On the one hand, the actuator delay of the CAV is typically constant and in the range of 0.3 to 0.6 seconds. On the other hand, the communication delay is in the range of 0.1-0.3 second and this changes stochastically based on the random nature of packet scheduling algorithms and the packet drops in wireless communication.

The stochastic process describing the motion of the CAV is characterized by its second moment dynamics. First, the solution of the stochastic system describing the dynamics of the CAV is partitioned into a deterministic part and into a stochastic part. Next, the second moment stability and the steady-state second moment of the stochastic part are investigated with the help of the method introduced above. Finally, it is shown how the CAV can benefit from applying the so-called delay matching to suppress the fluctuations induced by the stochastic delay while maintaining its robustness against the external noise presented by the CHV ahead.

The stochastically delayed differential equation describing the motion of the CAV can be written as

$$\begin{aligned}\dot{s}(t) &= v(t), \\ \dot{v}(t) &= \alpha(V(s_1(t - \tau_1(t)) - s(t - \tau(t)) - l) - v(t - \tau(t))) + \beta(v_1(t - \tau_1(t)) - v(t - \tau(t))).\end{aligned}\quad (65)$$

Here the dot stands for differentiation with respect to time t , s and s_1 denote the positions of the rear bumpers of the CAV and the CHV ahead, while v and v_1 denote their velocities, respectively; see Figure 7A. The length of the CAV is denoted with l and the headway is defined by

$$h = s_1 - s - l. \quad (66)$$

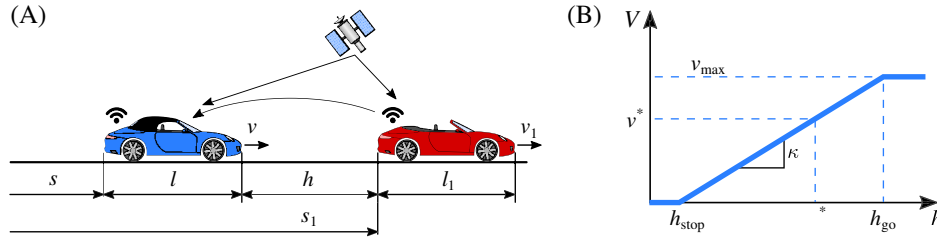


FIGURE 7 Connected car-following example: A, Sketch of a connected automated vehicle following a connected human-driven vehicle and, B, the range policy (67) [Colour figure can be viewed at wileyonlinelibrary.com]

In (65), the gains α and β are used to correct velocity errors, τ represents the actuator delay of the CAV, while τ_1 incorporates the communication delay as well as the actuator delay. The desired velocity is determined by the nonlinear (piecewise linear) range policy function

$$V(h) = \begin{cases} 0 & \text{if } h \leq h_{\text{st}}, \\ \kappa(h - h_{\text{st}}) & \text{if } h_{\text{st}} < h < h_{\text{go}}, \\ v_{\text{max}} & \text{if } h \geq h_{\text{go}}, \end{cases} \quad (67)$$

shown in Figure 7B, where $\kappa = v_{\text{max}} / (h_{\text{go}} - h_{\text{st}})$. That is, the desired velocity is zero for small headways ($h \leq h_{\text{st}}$) and equal to the speed limit v_{max} for large headways ($h \geq h_{\text{go}}$). Between these, the desired velocity increases with the headway linearly, with gradient κ . Note that when $h_{\text{st}} = 0$, the quantity $1/\kappa$ is often referred to as the time headway.

We consider that the CHV's velocity satisfy $\mathbb{E}(v_1(t)) = v^*$ which allows us to partition its position and velocity as

$$\begin{bmatrix} s_1(t) \\ v_1(t) \end{bmatrix} = \begin{bmatrix} v^* \\ 0 \end{bmatrix} t + \begin{bmatrix} s_1^* \\ v^* \end{bmatrix} + \begin{bmatrix} x_{1,s}(t) \\ x_{1,v}(t) \end{bmatrix}. \quad (68)$$

To describe the sum of the measurement error and the perturbation of the motion of the lead vehicle, the vector $\mathbf{x}_1 := [x_{1,s}, x_{1,v}]^\top$ is defined, that is, $\mathbb{E}(\mathbf{x}_1) = \mathbf{0}$. In the case of the CAV, the same partitioning leads to

$$\begin{bmatrix} s(t) \\ v(t) \end{bmatrix} = \begin{bmatrix} v^* \\ 0 \end{bmatrix} t + \begin{bmatrix} s^* \\ v^* \end{bmatrix} + \begin{bmatrix} x_s(t) \\ x_v(t) \end{bmatrix}, \quad (69)$$

where the vector $\mathbf{x} := [x_s, x_v]^\top$ collects the position and velocity perturbations of the CAV. Note that for connected cars, the holding time is always smaller than the minimum time delay of the system, that is, $T < \tau_1$, leading to $\mathbb{E}(\mathbf{x}(t)) = \mathbf{0}$.

Substituting the definitions of the positions and the velocities of the vehicles from (68) and (69) into (65), using the range policy (67), and taking the expected value of the resulting equation, the stationary average headway distance can be determined:

$$\lim_{t \rightarrow \infty} \mathbb{E}(h(t)) = \lim_{t \rightarrow \infty} \mathbb{E}(s_1(t) - s(t) - l) = s_1^* - s^* - l = V^{-1}(v^*) + (\bar{\tau}_1 - \bar{\tau})v^*. \quad (70)$$

Here $\bar{\tau}$ and $\bar{\tau}_1$ denote the average values of the delays and V^{-1} is only unique for $0 < v^* < v_{\text{max}}$; cf. (67). This shows that the average mismatch between the delays may result in an (undesired) increase of the stationary headway.

To characterize the quality of the CAV control in case of a dense, but continuously flowing traffic scenario ($h_{\text{stop}} < h(t) < h_{\text{go}}, \forall t$), the dynamics in the middle linear section of the range policy $V(h)$ need to be studied. This is described by the linear system

$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}\mathbf{x}(t - \tau(t)) + \mathbf{B}_1\mathbf{x}_1(t - \tau_1(t)) - ((\tau(t) - \bar{\tau})\mathbf{B} + (\tau_1(t) - \bar{\tau}_1)\mathbf{B}_1)\mathbf{v}_0, \quad (71)$$

where

$$\mathbf{A} = \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix}, \quad \mathbf{B} = \begin{bmatrix} 0 & 0 \\ -\alpha\kappa & -(\alpha + \beta) \end{bmatrix}, \quad \mathbf{B}_1 = \begin{bmatrix} 0 & 0 \\ \alpha\kappa & \beta \end{bmatrix}, \quad \mathbf{v}_0 = \begin{bmatrix} v^* \\ 0 \end{bmatrix}. \quad (72)$$

Note that in (71), the terms containing $\mathbf{x}_1(t)$ and \mathbf{v}_0 act as excitations on the system. Furthermore, it is assumed that the perturbation \mathbf{x} is small and the headway h does not leave the interval $[h_{st}, h_{go}]$.

6.1 | Effect of delay matching on connected cruise control

It was shown by (70) that a mismatch between the delays τ and τ_1 result in a shift from the desired stationary solution. Since the packets sent via V2V communication are time stamped, one may add some delay to the actuation delay to τ so that it matches τ_1 . Here we utilize the analytical techniques established above to evaluate the performance of the CAV when we apply vs. do not apply such delay matching.

In case of delay matching, the controller sets

$$\tau(t) = \tau_1(t), \quad (73)$$

and, since the vector \mathbf{v}_0 is in the nullspace of the matrix $\mathbf{B} + \mathbf{B}_1$, the linear system (71) simplifies to

$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}\mathbf{x}(t - \tau_1(t)) + \mathbf{B}_1\mathbf{x}_1(t - \tau_1(t)). \quad (74)$$

In case of no delay matching, the actuation delay is left constant, that is,

$$\tau(t) \equiv \tau = \bar{\tau}, \quad (75)$$

and (71) becomes

$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}\mathbf{x}(t - \tau) + \mathbf{B}_1\mathbf{x}_1(t - \tau_1(t)) - \mathbf{B}_1\mathbf{v}_0 \cdot (\tau_1(t) - \bar{\tau}_1), \quad (76)$$

with no stochasticity in the delay τ .

For simplicity, the case when the leading vehicle is moving with a constant speed is considered, that is, $\mathbf{x}_1(t) \equiv \mathbf{0}$. In this case, (74) has no added noise, while (76) is only excited by the delay noise. When presenting the results, we use the parameter $\kappa = 0.6[1/s]$, stationary velocity $v^* = 20[m/s]$, actuation delay $\tau = 0.5[s]$, holding time $T = 0.1[s]$. We also assume that the stochastic delay $\tau_1(t)$ can take values from the set $\tau_1(t) \in \{0.55[s], 0.65[s], 0.75[s]\}$ with probabilities $w_i \in \{0.25, 0.5, 0.25\}$, respectively. Note that with these parameters the delay matching not only causes the delay $\tau(t)$ to be stochastic, but it also increases its mean $\bar{\tau}$.

The stability charts are plotted in the plane of the control gains α and β in Figure 8A by utilizing (48). Note that the stability domain is smaller for the case with delay matching due to the fact that τ is increased in this case (unlike in case of, for example, suppressing chatter in machine tool vibrations via spindle speed variation, where the delay is varied around the original constant delay³⁰). However, since there is no added noise in this case, we obtain $\lim_{t \rightarrow \infty} x_s^2 = 0$ and $\lim_{t \rightarrow \infty} x_v^2 = 0$ for the stationary second moments. This is illustrated by the numerical simulation in Figure 8B corresponding to point P located at $(\alpha, \beta) = (0.4, 0.5)$ in the stability chart. The simulated trajectories (grey curve) approach zero quickly bringing the mean (thick green curve) and the standard deviation (thin green curve) to zero too.

When no delay matching is used, the stable domain is larger but, due to the delay-induced noise added to the system, the stationary second moment is not zero. The contours of $\lim_{t \rightarrow \infty} x_v^2$ are calculated using (58) are shown in Figure 8A. The numerical simulations in Figure 8C illustrate the dynamics for point P marked in the stability chart. The sample trajectory (grey curve) shows that the velocity of the CAV keeps changing in time so that the mean (thick blue curve) approaches zero while the standard deviation approaches the constant value calculated analytically.

Notice that while delay matching reduces the size of the stability domain, one may still have a large range of gain parameters to choose from. In particular, in the experimentally realistic range of $\alpha \in [0, 1], \beta \in [0, 1]$ stability is still ensured. Furthermore, the delay matching eliminates the unwanted stationary oscillations that lead to “jerky ride,” which typically has a negative effect on driver comfort and energy consumption. Note that in case of no delay matching, the delay-induced noise may be magnified to an extent, where (71) is not valid anymore (the range policy $V(h)$ saturates). However, in terms of control design, these parameter regions should be avoided, and instead of the stability boundary,

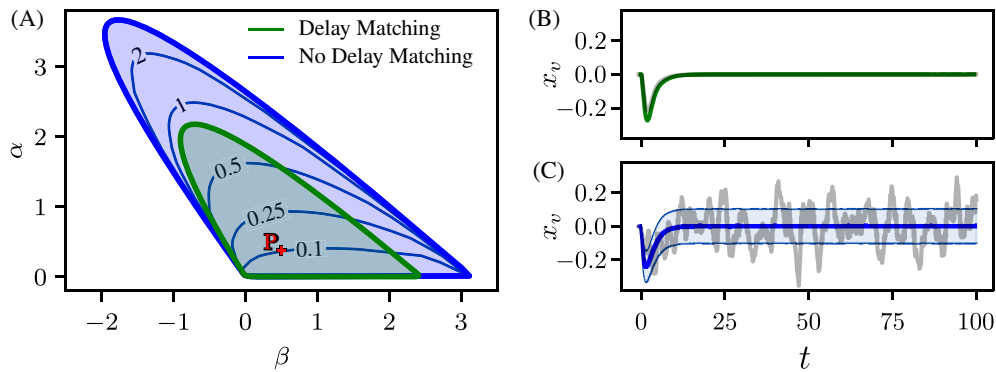


FIGURE 8 Performance of connected cruise control without added noise. A, Second moment stability charts with the stable parameter domains shaded. The lines with the numbers denote contours of stationary standard deviations of $x_v(t)$. B and C, The results of Monte Carlo simulations conducted for gain parameters $\alpha = 0.4$ and $\beta = 0.5$ (marked as P on the stability chart) the mean and the standard deviation are highlighted by the colored lines [Colour figure can be viewed at wileyonlinelibrary.com]

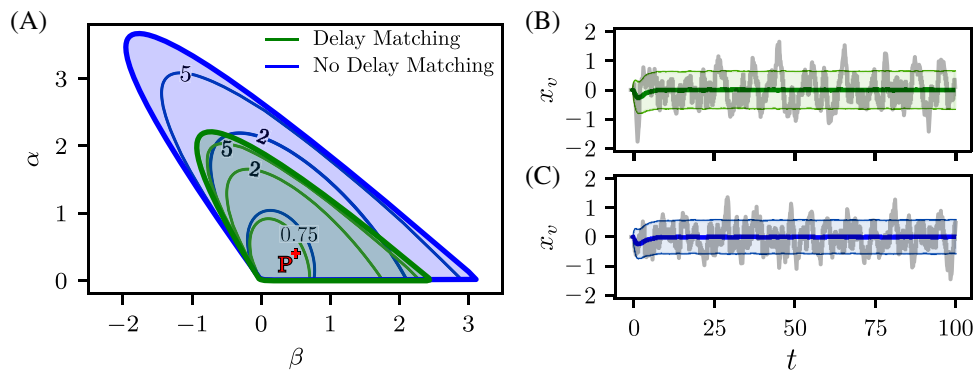


FIGURE 9 Performance of connected cruise control with added white noise. A, Second moment stability charts with the stable parameter domains shaded. The lines with the numbers denote contours of stationary standard deviations of $x_v(t)$. B and C, The results of Monte Carlo simulations conducted for gain parameters $\alpha = 0.4$ and $\beta = 0.5$ (marked as P on the stability chart) the mean and the standard deviation are highlighted by the colored lines [Colour figure can be viewed at wileyonlinelibrary.com]

the stationary second moment contours should be considered when choosing control parameters for the CAV to limit the amplitude of the perturbation \mathbf{x} .

The positive effects of delay matching can also be observed when the CHV ahead varies its velocity slowly (that is typical in real traffic situations).⁴⁴ In order to evaluate the effects for more severe motion perturbations, we model the perturbation dynamics of the CHV using white noise, that is,

$$\mathbf{x}_1(t - \tau_1(t)) := [\xi_s(t), \xi_v(t)]^\top, \quad (77)$$

where $\xi_s(t)$ and $\xi_v(t)$ are uncorrelated Gaussian white noise. This is indeed an over estimation of the severity of perturbations but provides a way to compare the behavior with and without delay matching. This leads to a noise excitation (49) with

$$\mathbf{\Omega} = \mathbf{B}_1, \quad \mathbf{\kappa} = \begin{cases} \mathbf{0}, & \text{in case of delay matching} \\ \mathbf{B}_1 \mathbf{v}_0, & \text{in case of no delay matching.} \end{cases} \quad (78)$$

The results are summarized in Figure 9 where panel (A) depicts the stability charts. The stability boundaries are identical to those in Figure 8A as these are still obtained by (48) while the contours calculated by (58) change due to the added white noise. Note that the contours obtained for the delayed matching case are quite similar to those obtained without delayed matching for small gain values, including the experimentally realistic range $\alpha \in [0, 1]$, $\beta \in [0, 1]$. For point P at $(\alpha, \beta) = (0.4, 0.5)$, this behavior is also illustrated by the numerical simulations shown in panels (B) and (C).

These demonstrate that when responding to severe perturbations using delay matching neither improves nor degrades the performance of the CAV.

To summarize this section, we demonstrated how the developed mathematical tools can be utilized for the control design of a connected vehicle system where stochastic packet loss results in stochasticity in the delay. Note that no external noise was added to the control system (65). However, some delay-induced additive noise was inherently present due to the packet drops (see Equations (68) to (72)), which had the same stochastic switching behavior as the stochastic delay. It was shown that this delay-induced additive term can be eliminated by delay-matching, that is, by deliberately delaying actuation of the CAV when a packet drop occurs. In (77) and (78), an additional white noise excitation, coming from the stochastic perturbation of the lead vehicle, was introduced, and it was shown that with delay-matching the controller maintains its robustness against external noise excitation.

7 | CONCLUSIONS

In this article, dynamical systems with stochastic delays and additive noise were investigated. The delay was assumed to jump between finitely many different values while remaining constant for a given holding time between the jumps. The additive noise was constructed from two different stochastic processes: one was an independent Gaussian white noise, called Wiener process, while the other was a process generated by the stochastically changing delay. A discrete time stochastic map was derived using semi-discretization and this was used to determine the dynamics of the first and second moments. Stability of the stationary moments was ensured by making the spectral radii of the corresponding coefficient matrices smaller than 1 and the stationary first and second moments were calculated.

The developed methods were applied to the stochastically delayed Hayes equation and it was demonstrated that the stationary first and second moments are periodic with period equal to the delay holding time. It was also shown that when the holding time was larger than the smallest possible time delay, the stationary first and second moments were not zero even when the noise exciting the system had zero mean. These results were also confirmed by statistical evaluation of Monte-Carlo simulations.

Finally, the established mathematical tools were used to support the claim, that matching the actuation delay (thus making it stochastic) in a connected vehicle system can improve the performance of CAVs. This occurs because delay matching eliminates the additive noise caused by the stochastically arriving information packets. It was shown that carefully choosing the control parameters one can preserve the robustness of the delay matched system even in the presence of a harsh white noise excitation.

ACKNOWLEDGMENTS

This work was funded by the Hungarian Scientific Research Fund (OTKA FK-124462 and PD-124646), by the Higher Education Excellence Program of the Ministry of Human Capacities in the frame of Artificial intelligence research area of Budapest University of Technology and Economics (BME FIKP-MI) and by the National Research, Development and Innovation Fund (TUDFO/51757/2019-ITM, Thematic Excellence Program, NKFIH ÚNKP-18-3-I-BME-160)

ORCID

Henrik T. Sykora  <https://orcid.org/0000-0003-1034-4387>

Dániel Bachrathy  <https://orcid.org/0000-0003-1491-1852>

Gábor Orosz  <https://orcid.org/0000-0002-9000-3736>

REFERENCES

1. Yaz T. The systems with delayed feedback. *Avtomatika i Telemekhanika*. 1928;7:107-129.
2. Orosz G, Wilson RE, Stépán G. Traffic jams: dynamics and control. *Philosoph Trans Royal Soc Lond A*. 2010;368(1928):4455-4479. <https://doi.org/10.1098/rsta.2010.0205>.
3. Volterra V. Sur la théorie mathématique des phénomènes héréditaires. *Journal de mathématiques pures et appliquées*. 1928;9:249-298.
4. Sadeghpour M, Orosz G. On the stability of continuous-time systems with stochastic delay: applications to gene regulatory circuits. 10th International Conference on Multibody Systems, Nonlinear Dynamics, and Control, vol 6, ASME; 2014. <https://doi.org/10.1115/DETC2014-35139>.
5. Tlustý J, Spacek L. *Self-Excited Vibrations on Machine Tools*. Prague, Czech Republic: Nakl. CSAV; 1954.
6. Altintas Y. *Manufacturing Automation: Metal Cutting Mechanics, Machine Tool Vibrations, and CNC Design*. 2nd ed. Cambridge, UK: Cambridge University Press; 2012.

7. Stépán G. Delay-differential equation models for machine tool chatter. Moon FC (Ed.), *Dynamics and Chaos in Manufacturing Processes*. New York: Wiley; 1998;165-192.
8. Stépán G. *Retarded Dynamical Systems: Stability and Characteristic Functions*. Research Notes in Mathematics Series. New York, NY: John Wiley & Sons, Inc; 1989.
9. Breda D, Maset S, Vermiglio R. *Stability of Linear Delay Differential Equations—A Numerical Approach with MATLAB*. New York, NY: Springer-Verlag; 2015.
10. Diekmann O, van Gils SA, Verduyn Lunel SM, Walther HO. Delay equations: functional- complex- and nonlinear analysis. *Applied Mathematical Sciences*. Vol 110. New York, NY: Springer; 1995.
11. Hale JK, Verduyn Lunel SM. *Introduction to Functional Differential Equations*. New York, NY: Springer-Verlag; 1993.
12. Kolmanovskii VB, Nosov VR. *Stability of Functional Differential Equations Mathematics in Science and Engineering*. Vol 180. London: Academic Press Inc; 1986.
13. Engelborghs K, Luzyanina T, Roose D. Numerical bifurcation analysis of delay differential equations using DDE-BIFTOOL. *ACM Trans Math Software*. 2002;28(1):1-21. <https://doi.org/10.1145/513001.513002>.
14. Kuske R. Competition of noise sources in systems with delay: the role of multiple time scales. *J Vib Control*. 2010;16(7-8):983-1003. <https://doi.org/10.1177/1077546309341104>.
15. Buckwar E, Kuske R, L'Esperance B, Soo T. Noise-sensitivity in machine tool vibrations. *Int J Bifurcat Chaos*. 2006;16(08):2407-2416. <https://doi.org/10.1142/S021812740601615X>.
16. Sykora HT, Bachrathy D, Stépán G. A theoretical investigation of the effect of the stochasticity in the material properties on the chatter detection during turning. Paper presented at: Proceedings of the International Design Engineering Technical Conferences and Computers and Information in Engineering Conference. American Society of Mechanical Engineers, Cleveland, OH; 2017.
17. Sykora HT, Bachrathy D, Stépán G. Stochastic semi-discretization for linear stochastic delay differential equations. *Int J Numer Methods Eng*. 2019;119(9):879-898. <https://doi.org/10.1002/nme.6076>.
18. Cloosterman MBG, van de Wouw N, Heemels WPMH, Nijmeijer H. Stability of networked control systems with uncertain time-varying delays. *IEEE Trans Automat Control*. 2009;54(7):1575-1580. <https://doi.org/10.1109/TAC.2009.2015543>.
19. Peters EGW, Marelli D, Quevedo DE, Fu M. Predictive control for networked systems affected by correlated packet loss. *Int J Robust Nonlinear Control*. 2019;29(15):5078-5094. <https://doi.org/10.1002/rnc.3896>.
20. Sadeghpour M, Breda D, Orosz G. Stability of linear continuous-time systems with stochastically switching delays. *IEEE Trans Automat Control*. 2019;64:4741-4741. <https://doi.org/10.1109/TAC.2019.2904491>.
21. Orosz G, Krauskopf B, Wilson RE. Traffic jam dynamics in a car-following model with reaction-time delay and stochasticity of drivers. *IFAC Proc Vol*. 2006;39(10):199-204. <https://doi.org/10.3182/20060710-3-IT-4901.00033>.
22. Qin WB, Gomez MM, Orosz G. Stability and frequency response under stochastic communication delays with applications to connected cruise control design. *IEEE Trans Intell Transp Syst*. 2017;18(2):388-403. <https://doi.org/10.1109/TITS.2016.2574246>.
23. Qin WB, Orosz G. Scalable stability analysis on large connected vehicle systems subject to stochastic communication delays. *Transp Res Part C Emerg Technol*. 2017;83:39-60. <https://doi.org/10.1016/j.trc.2017.07.005>.
24. Quevedo D, Silva E, Goodwin G. Control over unreliable networks affected by packet erasures and variable transmission delays. *IEEE J Select Areas Commun*. 2008;26:672-685. <https://doi.org/10.1109/JSAC.2008.080509>.
25. Krtolica R, Özgüner Ü, Chan H, Gökaş H, Winkelman J, Liubakka M. Stability of linear feedback systems with random communication delays. *Int J Control*. 1994;59(4):925-953.
26. Nilsson J, Bernhardsson B, Wittenmark B. Stochastic analysis and control of real-time systems with random time delays. *Automatica*. 1998;34(1):57-64.
27. Hu Z, Deng F. Robust H_∞ control for networked systems with transmission delays and successive packet dropouts under stochastic sampling. *Int J Robust Nonlinear Control*. 2017;27(1):84-107. <https://doi.org/10.1002/rnc.3559>.
28. Gomez MM, Sadeghpour M, Bennett MR, Orosz G, Murray RM. Stability of systems with stochastic delays and applications to genetic regulatory networks. *SIAM J Appl Dyn Syst*. 2016;15(4):1844-1873. <https://doi.org/10.1137/15m1031965>.
29. Gupta C, López JM, Azencott R, Bennett MR, Josić K, Ott W. Modeling delay in genetic networks: from delay birth-death processes to delay stochastic differential equations. *J Chem Phys*. 2014;140(20):204108. <https://doi.org/10.1063/1.4878662>.
30. Yilmaz A, AL-Regib E, Ni J. Machine tool chatter suppression by multi-level random spindle speed variation. *J Manufact Sci Eng*. 2002;124(2):208-216. <https://doi.org/10.1115/1.1378794>.
31. Kats I. On the stability of systems with random delay in the first approximation. *J Appl Math Mech*. 1967;31(3):447-452.
32. Lidskii EA. On stability of motion of a system with random delays. *Differentsial'nye Uravnenia*. 1965;1(1):96-101.
33. Kolmanovsky I, Maizenberg TL. Stochastic stability of a class of nonlinear systems with randomly varying time-delay. Paper presented at: Proceedings of the 2000 American Control Conference; vol. 6, 2000:4304-4308; Chicago, IL: ACC. <https://doi.org/10.1109/ACC.2000.877033>.
34. Wang J, Kuske R. The influence of parametric and external noise in act-and-wait control with delayed feedback. *Chaos Interdiscipl J Nonlinear Science*. 2017;27(11):114319. <https://doi.org/10.1063/1.5006776>.
35. Song R, Zhu Q. Stability of linear stochastic delay differential equations with infinite Markovian switchings. *Int J Robust Nonlinear Control*. 2018;28(3):825-837. <https://doi.org/10.1002/rnc.3905>.
36. Lu J, Xi Y, Li D. Stochastic model predictive control for probabilistically constrained Markovian jump linear systems with additive disturbance. *Int J Robust Nonlinear Control*. 2019;29(15):5002-5016. <https://doi.org/10.1002/rnc.3971>.

37. Seron MM, Goodwin GC, Carrasco DS. Stochastic model predictive control: insights and performance comparisons for linear systems. *Int J Robust Nonlinear Control*. 2019;29(15):5038-5057. <https://doi.org/10.1002/rnc.4106>.
38. Insperger T, Stépán G. *Semi-Discretization for Time-Delay Systems Applied Mathematical Sciences*. Vol 178. New York, NY: Springer; 2011.
39. Arnold L. *Stochastic Differential Equations: Theory and Applications*. Munich: R. Oldenbourg Verlag; 1973.
40. Bachrathy D, Stépán G. Bisection method in higher dimensions and the efficiency number. *Periodica Polytechnica Mech Eng*. 2012;56(2):81. <https://doi.org/10.3311/pp.me.2012-2.01>.
41. Depetri GI, Pereira FAC, Marin B, Baptista MS, Sartorelli JC. Dynamics of a parametrically excited simple pendulum. *Chaos Interdiscipl J Nonlinear Sci*. 2018;28(3):033103. <https://doi.org/10.1063/1.5016819>.
42. Orosz G. Connected cruise control: modelling, delay effects, and nonlinear behaviour. *Veh Syst Dyn*. 2016;54(8):1147-1176. <https://doi.org/10.1080/00423114.2016.1193209>.
43. Ge JI, Avedisov SS, He CR, Qin WB, Sadeghpour M, Orosz G. Experimental validation of connected automated vehicle design among human-driven vehicles. *Transp Res Part C*. 2018;91:335-352. <https://doi.org/10.1016/j.trc.2018.04.005>.
44. Beregi S, Avedisov SS, Takács D, He CR, Orosz G. On the resilience of connected automated vehicles. Preprint. 2020.
45. Laub AJ. *Matrix Analysis for Scientists and Engineers*. Thailand, Asia: SIAM; 2005.

How to cite this article: Sykora HT, Sadeghpour M, Ge JI, Bachrathy D, Orosz G. On the moment dynamics of stochastically delayed linear control systems. *Int J Robust Nonlinear Control*. 2020;30:8074-8097. <https://doi.org/10.1002/rnc.5218>

APPENDIX A. HIGHER ORDER SEMI-DISCRETIZATION METHOD

Using the methods in Reference 38, system (1) can be discretized to:

$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B} \left(\sum_{i_q=0}^q L_{i_q}^{(q)}(t) \mathbf{x}(n\Delta t - (r(n) - i_q)\Delta t) \right) + \boldsymbol{\gamma}(t), \quad (\text{A1})$$

where $t \in [n\Delta t, (n+1)\Delta t]$, $n \in \mathbb{N}$ and the time resolution is $\Delta t = T/l$ is given by the holding time T and $l \in \mathbb{N}$. Here the delayed state is approximated by a q th order Lagrange polynomial, where

$$L_{i_q}^{(q)}(t) = \prod_{j_q=0, j_q \neq i_q}^q \frac{t - \tau - (n + j_q - r(n))\Delta t}{(i_q - j_q)\Delta t}, \quad (\text{A2})$$

and

$$r(n) = \left\lfloor \frac{\tau(n\Delta t)}{\Delta t} + \frac{q}{2} \right\rfloor. \quad (\text{A3})$$

In the main body of the article, the zeroth order semi-discretization is used and indeed (A1), (A2), and (A3) give (1) and (4) when $q=0$.

Solving the differential equation (A1) on the interval $[n\Delta t, (n+1)\Delta t]$ leads to

$$\mathbf{x}((n+1)\Delta t) = e^{\mathbf{A}\Delta t} \mathbf{x}(n\Delta t) + \sum_{i_q=0}^q \left(\int_{n\Delta t}^{(n+1)\Delta t} e^{\mathbf{A}((n+1)\Delta t-s)} \mathbf{B} L_{i_q}^{(q)}(s) ds \mathbf{x}(n\Delta t - (r(n) - i_q)\Delta t) \right) + \int_{n\Delta t}^{(n+1)\Delta t} e^{\mathbf{A}((n+1)\Delta t-s)} \boldsymbol{\gamma}(s) ds. \quad (\text{A4})$$

APPENDIX B. DERIVATION OF THE SECOND MOMENT DYNAMICS

In this section, the derivation of the second moment dynamics is shown, namely, how

$$\bar{\mathbf{z}}(k+1) := \mathbb{E}(\mathbf{z}(k+1) \otimes \mathbf{z}(k+1)) = \mathbb{E}((\mathbf{G}(k)\mathbf{z}(k) + \mathbf{g}(k)) \otimes (\mathbf{G}(k)\mathbf{z}(k) + \mathbf{g}(k))), \quad (\text{B1})$$

reduces to the form defined in (19) with the terms in (20).

Utilizing the bilinearity and associativity of the Kronecker product, namely, $\mathbf{A} \otimes (\mathbf{B} + \mathbf{C}) = \mathbf{A} \otimes \mathbf{B} + \mathbf{A} \otimes \mathbf{C}$ ⁴⁵ and the linearity of the expected value, (B1) yields

$$\bar{\bar{\mathbf{z}}}(k+1) = \mathbb{E}(\mathbf{G}(k)\mathbf{z}(k) \otimes \mathbf{G}(k)\mathbf{z}(k)) + \mathbb{E}(\mathbf{G}(k)\mathbf{z}(k) \otimes \mathbf{g}(k)) + \mathbb{E}(\mathbf{g}(k)\mathbf{G}(k) \otimes \mathbf{z}(k)) + \mathbb{E}(\mathbf{g}(k) \otimes \mathbf{g}(k)). \quad (\text{B2})$$

Now applying the property $(\mathbf{AB}) \otimes (\mathbf{CD}) = (\mathbf{A} \otimes \mathbf{C})(\mathbf{B} \otimes \mathbf{D})$ to the first term of (B2) and the using independence of $\mathbf{G}(k)$ and $\mathbf{z}(k)$, Equation (B2) can be reduced to the form given in (19), namely,

$$\bar{\bar{\mathbf{z}}}(k+1) = \bar{\bar{\mathbf{G}}}\bar{\bar{\mathbf{z}}}(k) + \bar{\bar{\mathbf{c}}}_z(k) + \bar{\bar{\mathbf{g}}}(k), \quad (\text{B3})$$

where

$$\begin{aligned} \bar{\bar{\mathbf{G}}} &:= \mathbb{E}(\mathbf{G}(k) \otimes \mathbf{G}(k)), \\ \bar{\bar{\mathbf{c}}}_z(k) &:= \mathbb{E}(\mathbf{G}(k)\mathbf{z}(k) \otimes \mathbf{g}(k)) + \mathbb{E}(\mathbf{g}(k) \otimes \mathbf{G}(k)\mathbf{z}(k)), \\ \bar{\bar{\mathbf{g}}}(k) &:= \mathbb{E}(\mathbf{g}(k) \otimes \mathbf{g}(k)). \end{aligned} \quad (\text{B4})$$

Next, the term $\bar{\bar{\mathbf{c}}}_z(k)$ is further simplified using the definition of the Kronecker product of two vectors \mathbf{a} and \mathbf{b} of the same length, namely, $\mathbf{a} \otimes \mathbf{b} = \text{vec}(\mathbf{ba}^\top)$ and the properties of the vectorization $\text{vec}(\mathbf{ABC}) = (\mathbf{C}^\top \otimes \mathbf{A})\text{vec}(\mathbf{B})$ where $\text{vec}(\mathbf{a}) = \text{vec}(\mathbf{a}^\top) = \mathbf{a}$ ⁴⁵ resulting in

$$\begin{aligned} \mathbb{E}(\mathbf{G}(k)\mathbf{z}(k) \otimes \mathbf{g}(k)) &= \mathbb{E}(\text{vec}(\mathbf{g}(k)\mathbf{z}(k)^\top \mathbf{G}(k)^\top)) = \mathbb{E}((\mathbf{G}(k) \otimes \mathbf{g}(k))\mathbf{z}(k)), \\ \mathbb{E}(\mathbf{g}(k) \otimes \mathbf{G}(k)\mathbf{z}(k)) &= \mathbb{E}(\text{vec}(\mathbf{G}(k)\mathbf{z}(k)\mathbf{g}(k)^\top)) = \mathbb{E}((\mathbf{g}(k) \otimes \mathbf{G}(k))\mathbf{z}(k)). \end{aligned} \quad (\text{B5})$$

Substituting these terms into (B4), the form of $\bar{\bar{\mathbf{c}}}_z(k)$ defined in (20) is obtained, namely,

$$\bar{\bar{\mathbf{c}}}_z(k) := \mathbb{E}((\mathbf{g}(k) \otimes \mathbf{G}(k) + \mathbf{G}(k) \otimes \mathbf{g}(k))\mathbf{z}(k)). \quad (\text{B6})$$

APPENDIX C. DERIVATION OF THE MEAN AND SECOND MOMENT DYNAMICS WITH GAUSSIAN WHITE NOISE AND DELAY-INDUCED NOISE

C.1 Derivation of mean dynamics

Here we want to show that when the additive noise is given by (49), the mean dynamics (16) reduce to

$$\bar{\mathbf{z}}(k+1) = \bar{\mathbf{G}}\bar{\mathbf{z}}(k) + \bar{\mathbf{g}}, \quad (\text{C1})$$

and we want to derive the disturbance term $\bar{\mathbf{g}}$; see (50) to (52). Let us first repeat the disturbance term from (16) here

$$\bar{\mathbf{g}}(k) = \sum_{m=0}^{l-1} \mathbb{E}(\mathbf{F}(kl)^{l-1-m} \mathbf{f}(kl+m)). \quad (\text{C2})$$

Note that $\mathbf{F}(kl)$ only depends on the delay $\tau(t)$ in the interval $(kT, (k+1)T]$ while $\mathbf{f}(kl+m)$, $m=0, 1, \dots, l-1$ depends both on the delay-in the interval $(kT, (k+1)T]$ and on the Gaussian noise $\xi(t)$ in the interval $(kT + m\Delta t, kT + (m+1)\Delta t]$. The interval $(kT, (k+1)T]$ is denoted by I_k for simplicity of notation in the rest of this section. Now (C2) is expanded as

$$\begin{aligned} \bar{\mathbf{g}}(k) &= \sum_{m=0}^{l-1} \sum_{j=1}^J \mathbb{E}(\mathbf{F}(kl)^{l-1-m} \mathbf{f}(kl+m) | \tau(t) = \tau_j, t \in I_k) \mathbb{P}(\tau(t) = \tau_j, t \in I_k) \\ &= \sum_{m=0}^{l-1} \sum_{j=1}^J w_j \mathbf{F}_j^{l-1-m} \mathbb{E}(\mathbf{f}(kl+m) | \tau(t) = \tau_j, t \in I_k). \end{aligned} \quad (\text{C3})$$

Consider the last term above. In view of (9) and (10), we have

$$\begin{aligned} & \mathbb{E}(\mathbf{w}_\gamma(kl+m)|\tau(t) = \tau_j, t \in I_k) \\ &= \mathbb{E} \left(\int_{(kl+m)\Delta t}^{(kl+m+1)\Delta t} e^{\mathbf{A}((kl+m+1)\Delta t-s)} (\mathbf{\Omega} d\mathbf{W}_s + \boldsymbol{\kappa}(\tau(s) - \bar{\tau}) ds) | \tau(t) = \tau_j, t \in I_k \right) \\ &= \mathbb{E} \left(\int_{(kl+m)\Delta t}^{(kl+m+1)\Delta t} e^{\mathbf{A}((kl+m+1)\Delta t-s)} \mathbf{\Omega} d\mathbf{W}_s \right) + \mathbb{E} \left(\int_{(kl+m)\Delta t}^{(kl+m+1)\Delta t} e^{\mathbf{A}((kl+m+1)\Delta t-s)} \boldsymbol{\kappa}(\tau_j - \bar{\tau}) ds \right). \end{aligned} \quad (\text{C4})$$

In view of (10), we have $\mathbb{E} \left(\int_0^{\Delta t} e^{\mathbf{A}(\Delta t-s)} \mathbf{\Omega} d\mathbf{W}_s \right) = \mathbf{0}$, and therefore, (C4) reduces to

$$\mathbb{E}(\mathbf{w}_\gamma(kl+m)|\tau(t) = \tau_j, t \in I_k) = (e^{\mathbf{A}\Delta t} - \mathbf{I})\mathbf{A}^{-1}\boldsymbol{\kappa}(\tau_j - \bar{\tau}), \quad (\text{C5})$$

assuming that \mathbf{A} is invertible. Substituting (C5) in (C3), we get

$$\begin{aligned} \bar{\mathbf{g}}(k) &= \sum_{m=0}^{l-1} \sum_{j=1}^J w_j \mathbf{F}_j^{l-1-m} \bar{\mathbf{f}}_j \\ &= \sum_{j=1}^J w_j \hat{\mathbf{F}}_j \bar{\mathbf{f}}_j \\ &:= \bar{\mathbf{g}}, \end{aligned} \quad (\text{C6})$$

where $\bar{\mathbf{f}}_j$ and $\hat{\mathbf{F}}_j$ are given in (52).

C.2 Derivation of second moment dynamics

Recall the second moment dynamics given by (19). Here (19) is simplified to the form

$$\bar{\bar{\mathbf{z}}}(k+1) = \bar{\bar{\mathbf{G}}}\bar{\bar{\mathbf{z}}}(k) + \bar{\bar{\mathbf{H}}}\bar{\bar{\mathbf{z}}}(k) + \bar{\bar{\mathbf{g}}}_2, \quad (\text{C7})$$

and obtain the terms $\bar{\bar{\mathbf{H}}}$ and $\bar{\bar{\mathbf{g}}}_2$ when the noise is given by (49); see (53), (55), and (56). Recall that $\mathbf{z}(k)$ is independent of $\mathbf{g}(k)$ and $\mathbf{G}(k)$ as mentioned before. Therefore, from (20) we have

$$\bar{\bar{\mathbf{c}}}_z(k) = \mathbb{E}(\mathbf{g}(k) \otimes \mathbf{G}(k) + \mathbf{G}(k) \otimes \mathbf{g}(k))\mathbb{E}(\mathbf{z}(k)). \quad (\text{C8})$$

Now observe that

$$\begin{aligned} \mathbb{E}(\mathbf{g}(k) \otimes \mathbf{G}(k)) &= \mathbb{E} \left(\left(\sum_{m=0}^{l-1} \mathbf{F}(kl)^{l-1-m} \mathbf{f}(kl+m) \right) \otimes \mathbf{G}(k) \right) \\ &= \sum_{j=1}^J w_j \mathbb{E} \left(\left(\sum_{m=0}^{l-1} \mathbf{F}(kl)^{l-1-m} \mathbf{f}(kl+m) \right) \otimes \mathbf{G}(k) | \tau(t) = \tau_j, t \in I_k \right) \\ &= \sum_{j=1}^J w_j \left(\sum_{m=0}^{l-1} \mathbf{F}_j^{l-1-m} \mathbb{E}(\mathbf{f}(kl+m) | \tau(t) = \tau_j, t \in I_k) \right) \otimes \mathbf{F}_j^l \\ &= \sum_{j=1}^J w_j \left(\sum_{m=0}^{l-1} \mathbf{F}_j^{l-1-m} \bar{\mathbf{f}}_j \right) \otimes \mathbf{F}_j^l \\ &= \sum_{j=1}^J w_j \hat{\mathbf{F}}_j \bar{\mathbf{f}}_j \otimes \mathbf{F}_j^l, \end{aligned} \quad (\text{C9})$$

cf. (52). Now, from (C8) and (C9), we have

$$\bar{\bar{\mathbf{c}}}_z(k) = \bar{\bar{\mathbf{H}}}\bar{\mathbf{z}}(k), \quad (\text{C10})$$

where

$$\bar{\bar{\mathbf{H}}} := \sum_{j=1}^J w_j (\hat{\mathbf{F}}_j \bar{\mathbf{f}}_j \otimes \mathbf{F}_j^l + \mathbf{F}_j^l \otimes \hat{\mathbf{F}}_j \bar{\mathbf{f}}_j). \quad (\text{C11})$$

Let us now consider the last term in (19), that is, $\bar{\bar{\mathbf{g}}}(k)$. We have that

$$\begin{aligned} \bar{\bar{\mathbf{g}}}(k) &= \mathbb{E}(\mathbf{g}(k) \otimes \mathbf{g}(k)) \\ &= \sum_{j=1}^J w_j \mathbb{E}(\mathbf{g}(k) \otimes \mathbf{g}(k) | \tau(t) = \tau_j, t \in I_k) \\ &= \sum_{j=1}^J w_j \mathbb{E} \left(\left(\sum_{m=0}^{l-1} \mathbf{F}_j^{l-1-m} \mathbf{f}(kl+m) \right) \otimes \left(\sum_{m=0}^{l-1} \mathbf{F}_j^{l-1-m} \mathbf{f}(kl+m) \right) | \tau(t) = \tau_j, t \in I_k \right). \end{aligned} \quad (\text{C12})$$

Note that, given $\tau(t) = \tau_j, t \in I_k$, we can write $\mathbf{f}(kl+m)$ as

$$\mathbf{f}(kl+m) = \begin{bmatrix} \mathbf{w}_\gamma(kl+m) \\ \mathbf{0} \\ \vdots \\ \mathbf{0} \end{bmatrix} = \bar{\mathbf{f}}_j + \mathbf{f}_{w,m}, \quad (\text{C13})$$

in view of (10), (37), (49), and (52).

Substituting (C13) into (C12) leads to

$$\begin{aligned} \bar{\bar{\mathbf{g}}}(k) &= \sum_{j=1}^J w_j \mathbb{E} \left(\left(\sum_{m=0}^{l-1} \mathbf{F}_j^{l-1-m} \right) \bar{\mathbf{f}}_j \otimes \left(\sum_{m=0}^{l-1} \mathbf{F}_j^{l-1-m} \right) \bar{\mathbf{f}}_j + \left(\sum_{m=0}^{l-1} \mathbf{F}_j^{l-1-m} \mathbf{f}_{w,m} \right) \otimes \left(\sum_{m=0}^{l-1} \mathbf{F}_j^{l-1-m} \mathbf{f}_{w,m} \right) \right. \\ &\quad \left. + \left(\sum_{m=0}^{l-1} \mathbf{F}_j^{l-1-m} \right) \bar{\mathbf{f}}_j \otimes \left(\sum_{m=0}^{l-1} \mathbf{F}_j^{l-1-m} \mathbf{f}_{w,m} \right) + \left(\sum_{m=0}^{l-1} \mathbf{F}_j^{l-1-m} \mathbf{f}_{w,m} \right) \otimes \left(\sum_{m=0}^{l-1} \mathbf{F}_j^{l-1-m} \right) \bar{\mathbf{f}}_j \right). \end{aligned} \quad (\text{C14})$$

The last two terms in (C14) are zero, because $\mathbb{E}(\mathbf{f}_{w,m}) = \mathbf{0}$, in view of (37). Therefore,

$$\begin{aligned} \bar{\bar{\mathbf{g}}}(k) &= \sum_{j=1}^J w_j \hat{\mathbf{F}}_j \bar{\mathbf{f}}_j \otimes \hat{\mathbf{F}}_j \bar{\mathbf{f}}_j + \sum_{j=1}^J w_j \mathbb{E} \left(\sum_{m=0}^{l-1} \sum_{m'=0}^{l-1} \mathbf{F}_j^{l-1-m} \mathbf{f}_{w,m} \otimes \mathbf{F}_j^{l-1-m'} \mathbf{f}_{w,m'} \right) \\ &= \sum_{j=1}^J w_j (\hat{\mathbf{F}}_j \otimes \hat{\mathbf{F}}_j) (\bar{\mathbf{f}}_j \otimes \bar{\mathbf{f}}_j) + \sum_{j=1}^J w_j \mathbb{E} \left(\sum_{m=0}^{l-1} \sum_{m'=0}^{l-1} (\mathbf{F}_j^{l-1-m} \otimes \mathbf{F}_j^{l-1-m'}) (\mathbf{f}_{w,m} \otimes \mathbf{f}_{w,m'}) \right) \\ &= \sum_{j=1}^J w_j \left[(\hat{\mathbf{F}}_j \otimes \hat{\mathbf{F}}_j) (\bar{\mathbf{f}}_j \otimes \bar{\mathbf{f}}_j) + \sum_{m=0}^{l-1} \sum_{m'=0}^{l-1} (\mathbf{F}_j^{l-1-m} \otimes \mathbf{F}_j^{l-1-m'}) \mathbb{E}(\mathbf{f}_{w,m} \otimes \mathbf{f}_{w,m'}) \right]. \end{aligned} \quad (\text{C15})$$

Using (39), we obtain

$$\begin{aligned} \bar{\bar{\mathbf{g}}}(k) &= \sum_{j=1}^J w_j \left[(\hat{\mathbf{F}}_j \otimes \hat{\mathbf{F}}_j) (\bar{\mathbf{f}}_j \otimes \bar{\mathbf{f}}_j) + \sum_{m=0}^{l-1} (\mathbf{F}_j^{l-1-m} \otimes \mathbf{F}_j^{l-1-m}) \mathbb{E}(\mathbf{f}_{w,m} \otimes \mathbf{f}_{w,m}) \right] \\ &:= \bar{\bar{\mathbf{g}}}_2. \end{aligned} \quad (\text{C16})$$

This can be written in the form

$$\begin{aligned}\bar{\bar{\mathbf{g}}}_2 &= \sum_{j=1}^J w_j (\hat{\mathbf{F}}_j \otimes \hat{\mathbf{F}}_j) (\bar{\mathbf{f}}_j \otimes \bar{\mathbf{f}}_j) + \left(\sum_{j=1}^J w_j \hat{\mathbf{F}}_j \right) \bar{\bar{\mathbf{f}}}_w \\ &= \sum_{j=1}^J w_j (\hat{\mathbf{F}}_j \otimes \hat{\mathbf{F}}_j) (\bar{\mathbf{f}}_j \otimes \bar{\mathbf{f}}_j) + \bar{\bar{\mathbf{g}}},\end{aligned}\tag{C17}$$

using the definitions (45), where $\bar{\bar{\mathbf{g}}}$ is given in (46). Finally, the second moment dynamics can be obtained by substituting (C10) and (C17) into (19) which results in (C7).